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Session ID: **ENG445DK**



ENG-445

Building Energetics

**Cooling in Buildings –
Emission Systems and
Thermal Comfort**

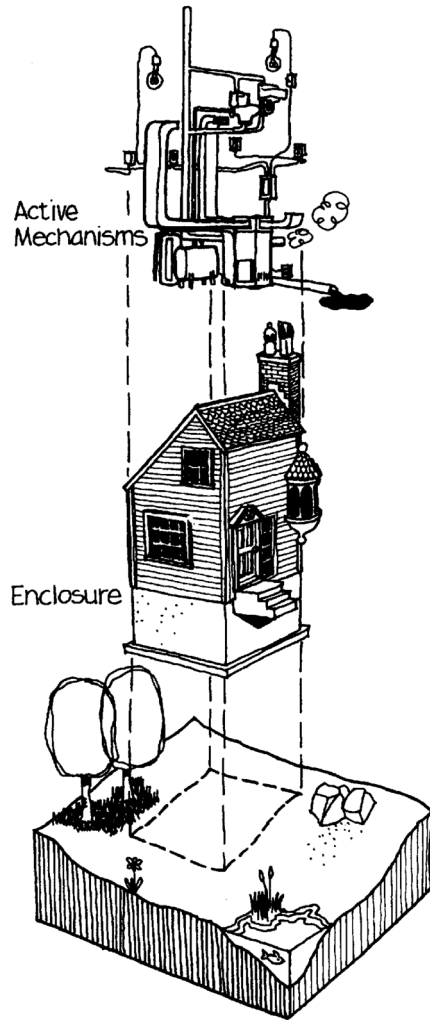
Assist. Professor
Dolaana Khovalyg

28 November 2024

Week	Date	Topic	Timing	Teacher	Project (AS, MF)
9	07/11	Building envelope, thermal performance of building elements	45' x 2	DK	Tutorial building envelope
		Exercises	45'		
10	14/11	Heating and cooling demand in buildings	45' x 2	DK	Free work
		Exercises	45'		
11	21/11	Thermal systems for heating and their effect of human comfort	45' x 2	JY	Free work
		Exercises	45'		
12	28/11	Thermal systems for cooling and their effect of human comfort	45' x 2	DK	Tutorial heating systems (emission systems)
		Exercises	45'		



JY – Jaafar Younes, a postdoc from the ICE lab



Source: Edward Allen «How Buildings Work» (2005)

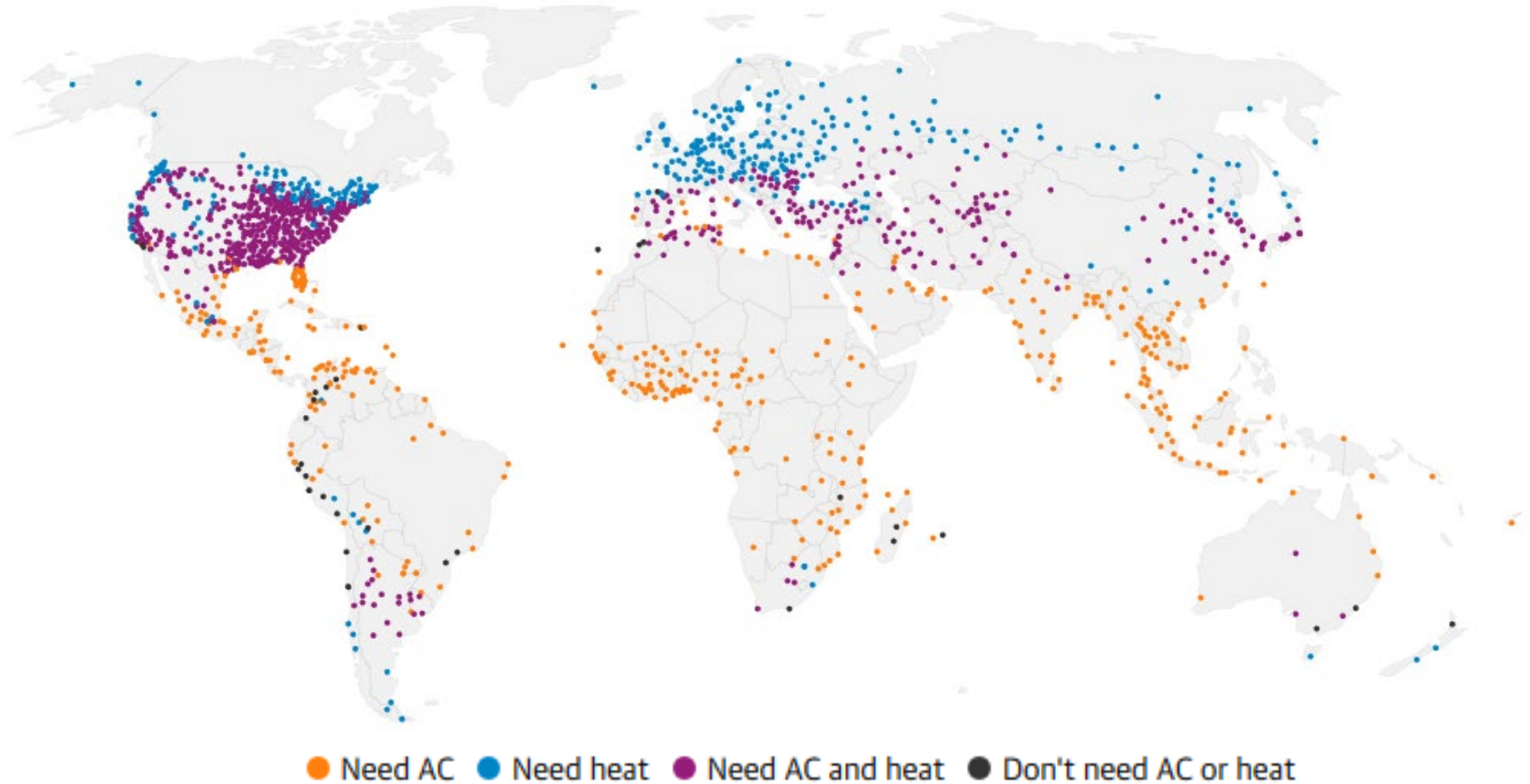
CONTENT:

- Introduction
- Passive Cooling Strategies
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 - Limiting heat flux
 - Radiant ceiling panels
 - Evaporative cooling
 - Dehumidification
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EPFL Indoor Thermal Conditioning: Overview

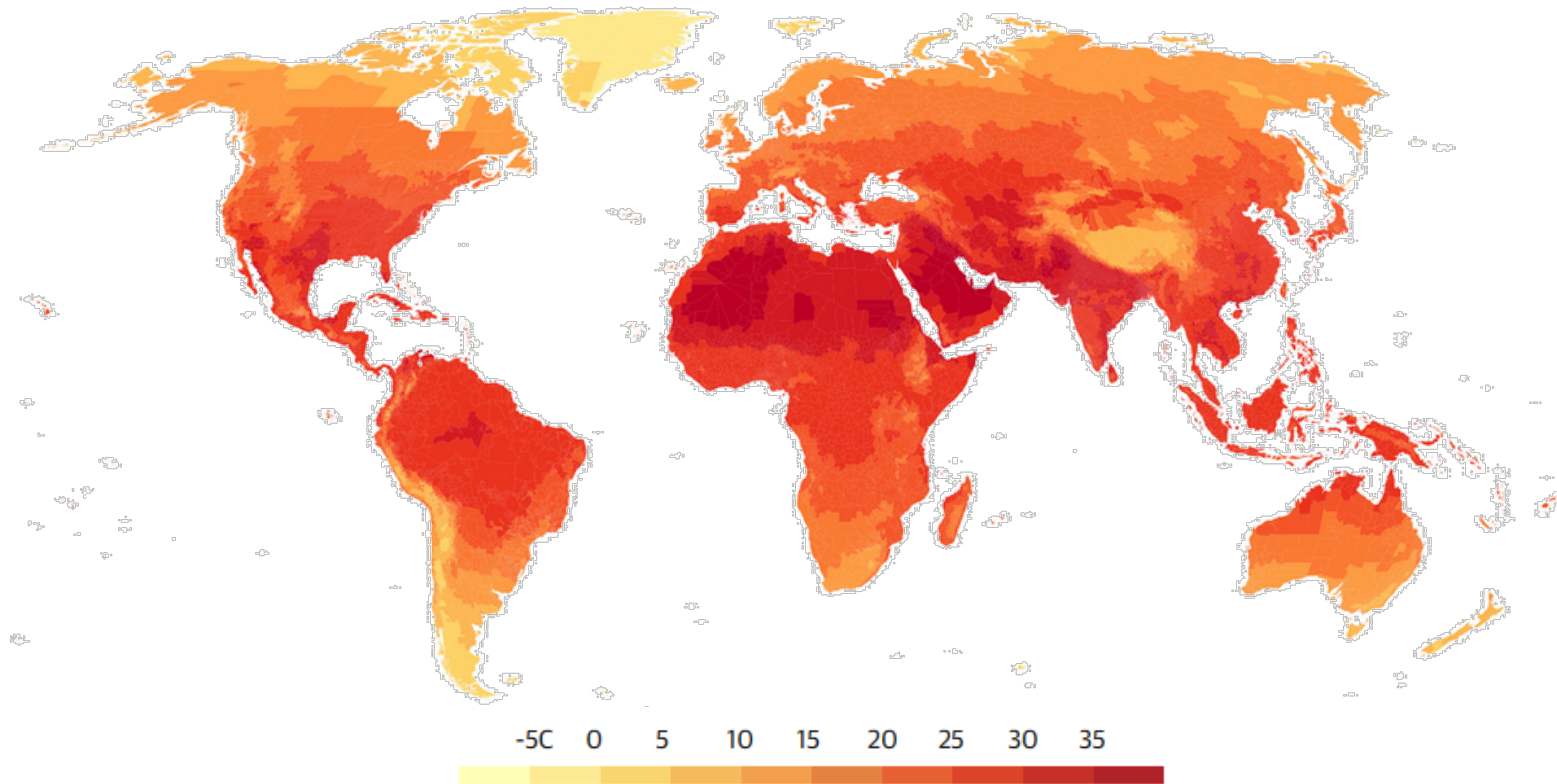
4

Assist. Prof. Dolaana Khovalyg



Source: <https://www.theguardian.com>

Average temperature between June and August, **2040-2059**



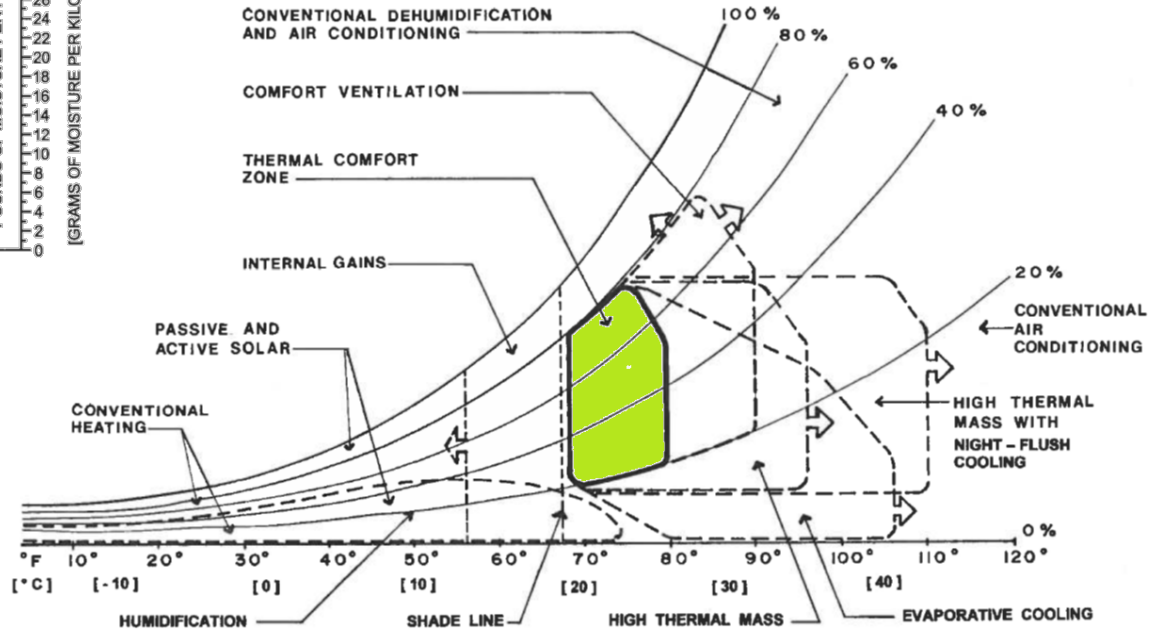
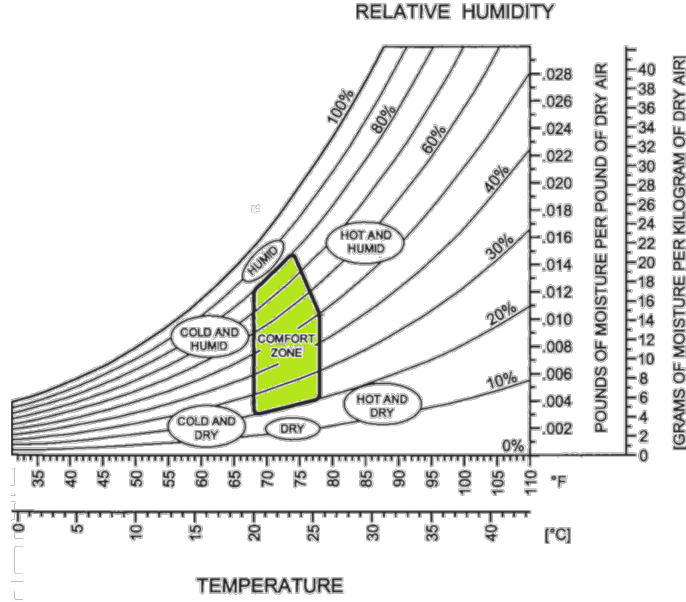
EPFL Indoor Cooling: Increasing Needs

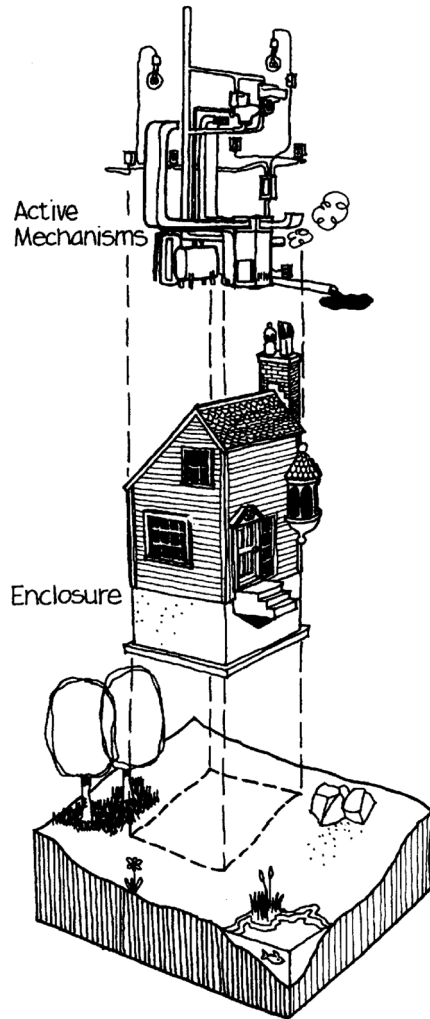


Source: [webpage](#)



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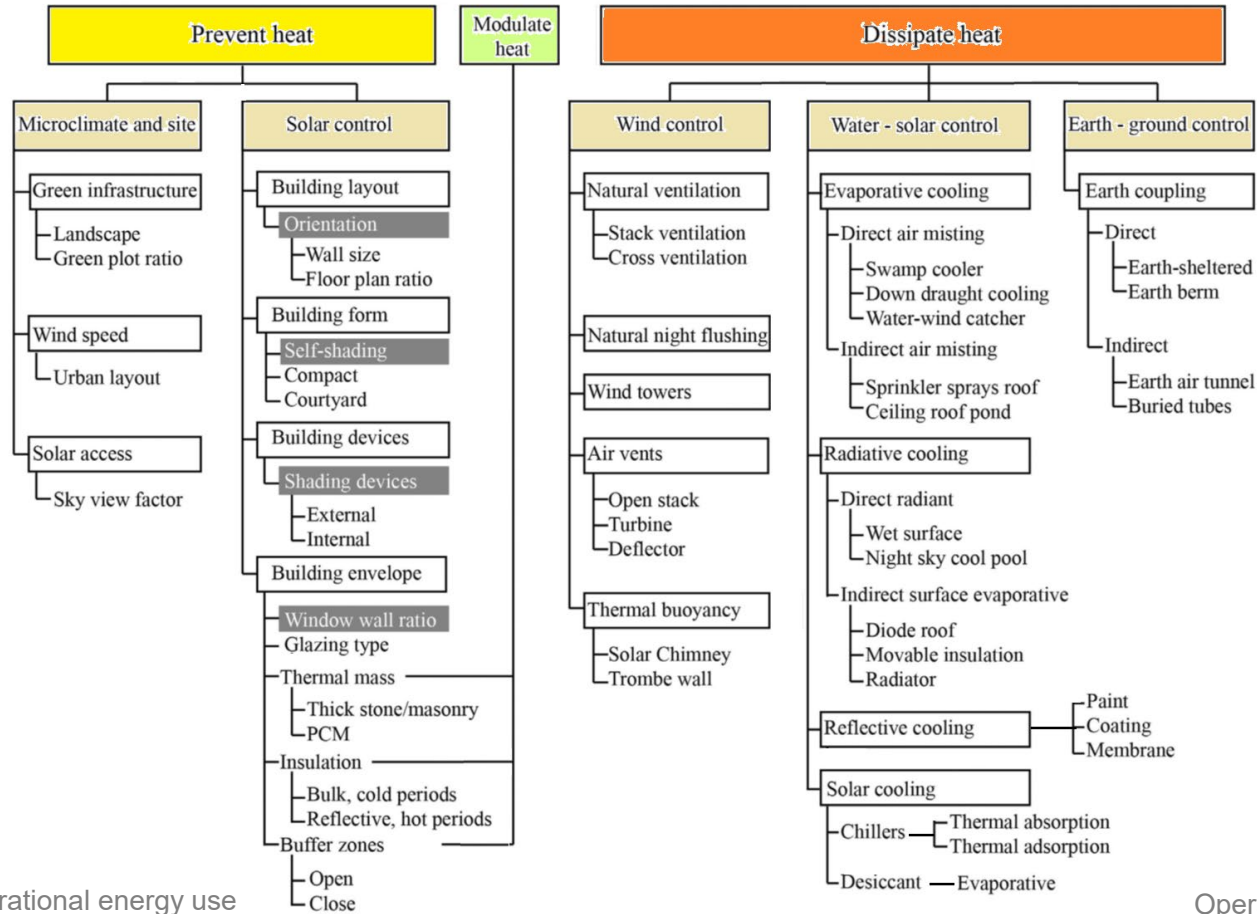




Source: Edward Allen «How Buildings Work» (2005)

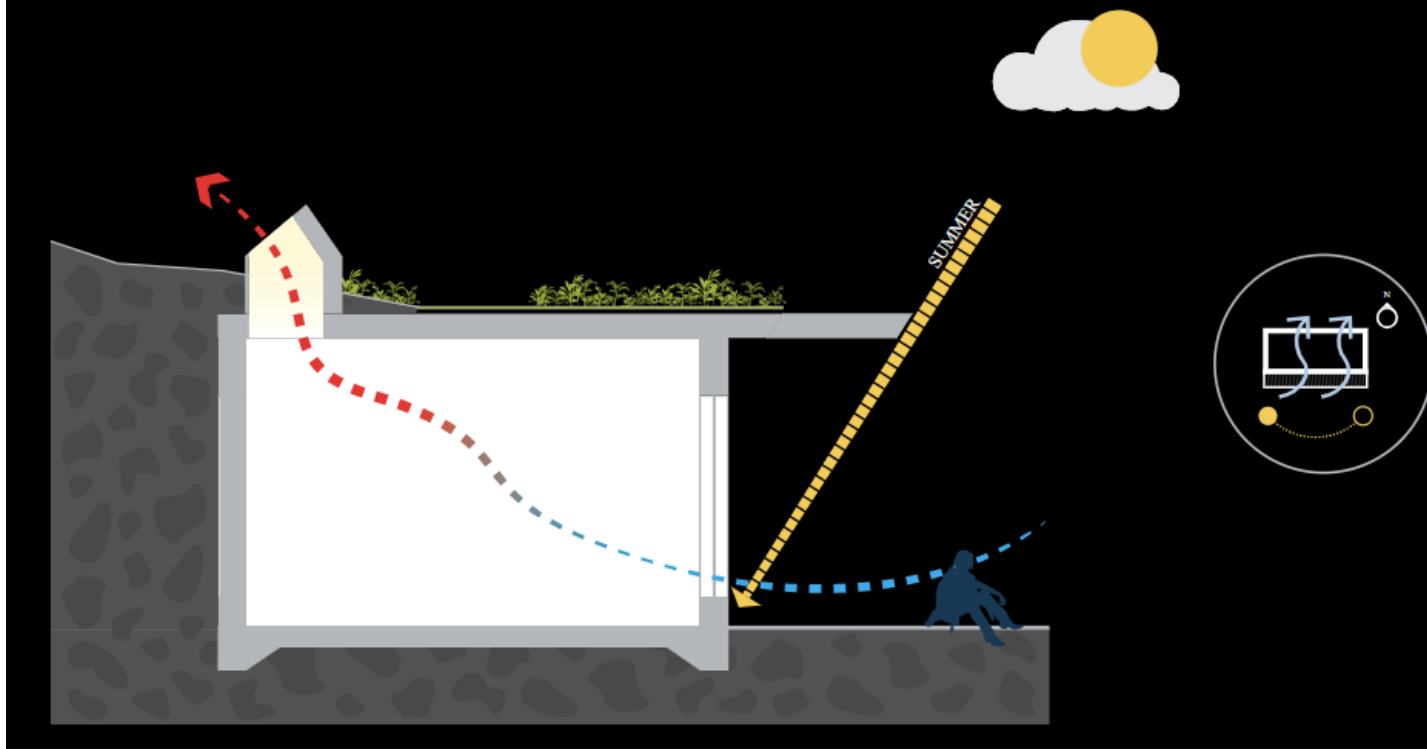
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EPFL Passive Cooling Strategies: **Demonstration**

<https://climatescout.arcadis.com/>



EPFL Passive Cooling Strategies: **Demonstration**

<https://climatescout.arcadis.com/>



Form for Cooling



Cool Roof



Green Roof



East/West Shading



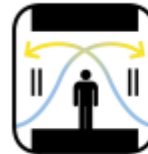
Solar Shading



Shading Devices



Stack Ventilation



Cross Ventilation



Night Vent Cooling



Building Facades



Earth Sheltering



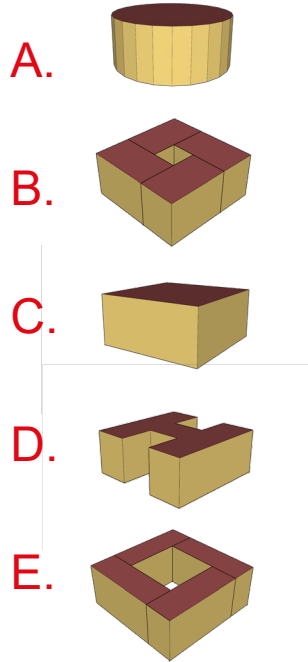
Evaporative Cooling
Towers

What kind of building shape is preferable for warm locations?

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What are the conditions to keep buildings cool using the thermal mass?

Discuss this question with your neighbors

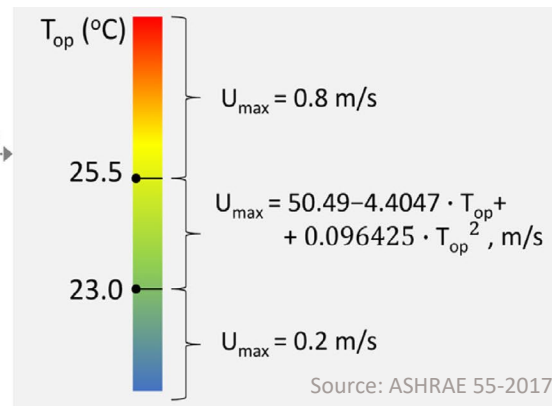
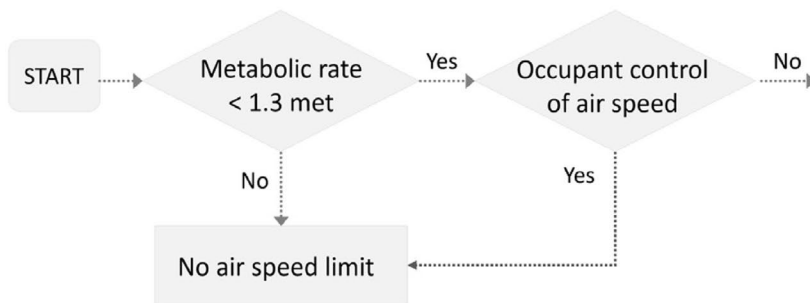
- A. Sufficient diurnal outdoor temperature fluctuations
- B. Use concrete in building structure
- C. Insulate building envelope
- D. Avoid heat gains during the day
- E. Enable nighttime flashing
- F. All above

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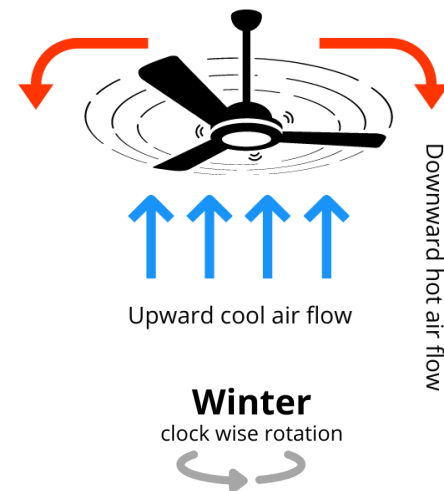
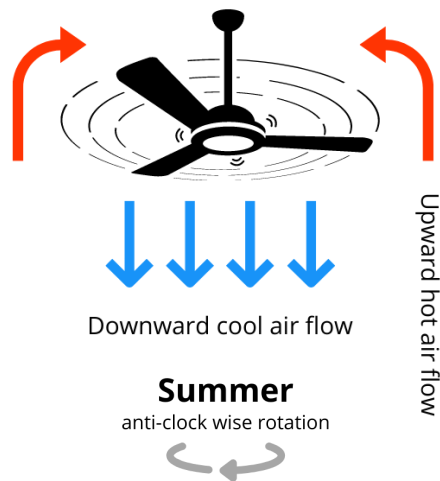
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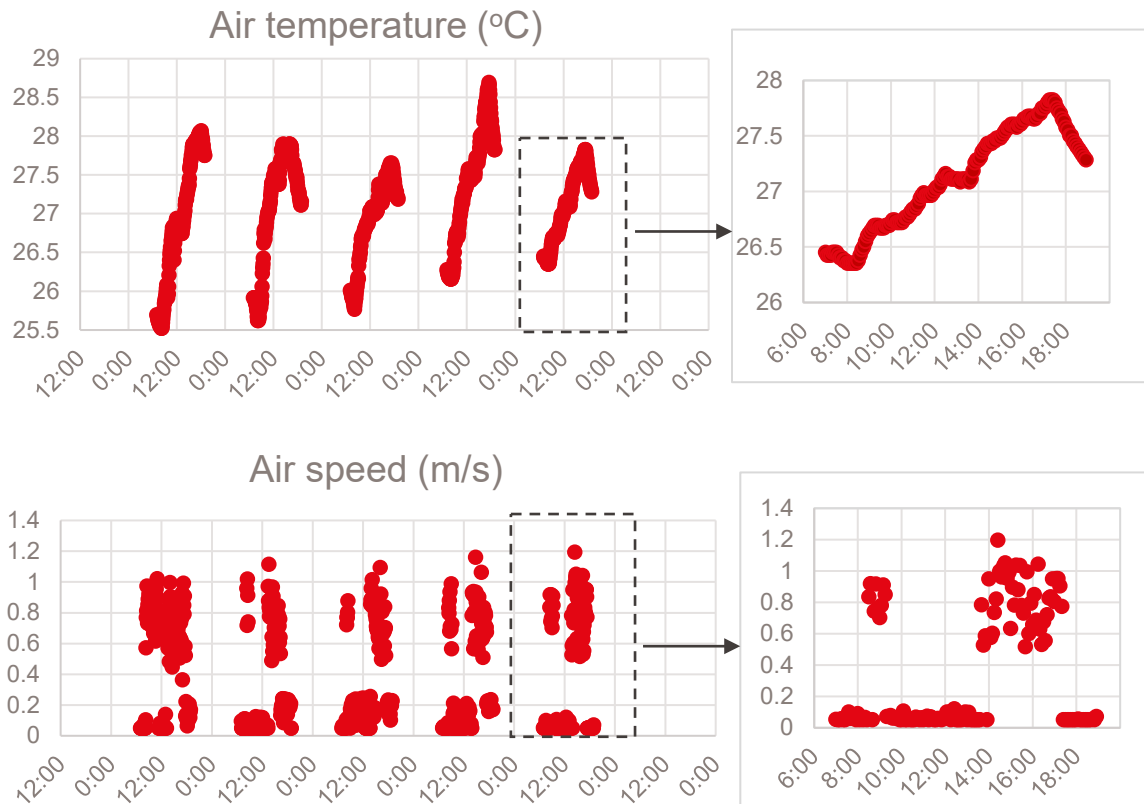
1. Choose **the building envelope configuration** that **reduces** heat gains
2. Reduce solar heat gains:
 - by choosing **clear colors/reflective surfaces** for external opaque surfaces
 - by **shading** the building with vegetation
 - by planning **windows with reasonable sizes** according to their orientations
 - by planning efficient **solar protections for fenestration**
3. Reduce **internal heat gains** from **indoor electric appliances**
4. Precisely define **thermally conditioned zones** and **position the thermal boundary** accordingly
5. Ensure *enough indoor elements* with **high thermal inertia** within thermal boundary zone in order **to reduce instantaneous overheating** the space
6. Remove the *excess heat* from the building using *passive means* (e.g., deploy **night-time cooling using natural ventilation** by planning openings)
7. Introduce simple cooling systems (e.g., **fans, evaporative cooling**)
8. Consider **active cooling *only to deal with overheated periods*** that cannot be reasonably avoided after applying steps (1)-(7)



Source: reviewsfairpurifiers.com



- Example of **elevated air speed** created by **a person** to improve the comfort
- Office monitoring in Geneva, summer 2020 (**eCOMBINE** project)
- **2 weeks** of measurements
- **1/30** person was using a desk fan
- Maximum air speed was between **0.6-1.2 m/s**



- Fans elevate air flow around the human and increase convective heat removal from the skin
- Increased convection helps also to increase latent heat removal (via evaporation)
- Sensation scale correction up to 1 point (from PMV= +0.43 to PVM= - 0.48)

Operative temperature

28 °C

Air speed

0.2 m/s

Relative humidity

50 %

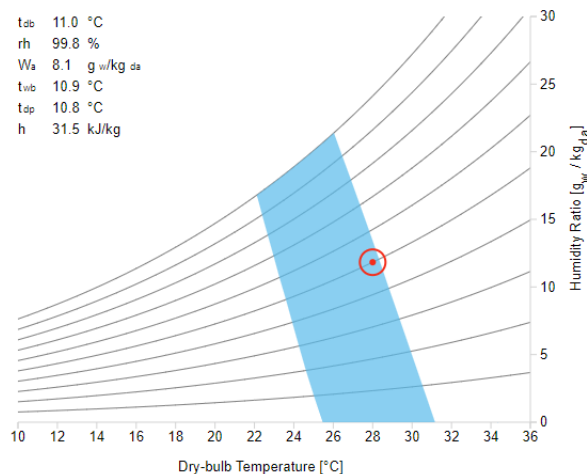
Metabolic rate

1 met

Clothing level

0.5 clo

t_{ab} 11.0 °C
 rh 99.8 %
 W_a 8.1 g w/kg da
 t_{wb} 10.9 °C
 t_{dp} 10.8 °C
 h 31.5 kJ/kg



Operative temperature

28 °C

Air speed

1 m/s

Relative humidity

50 %

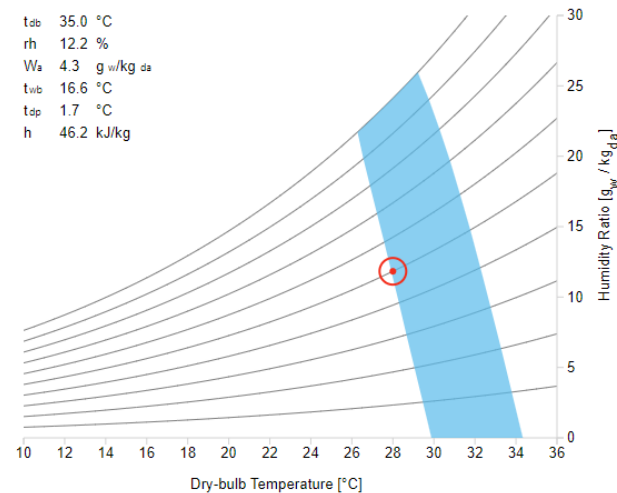
Metabolic rate

1 met

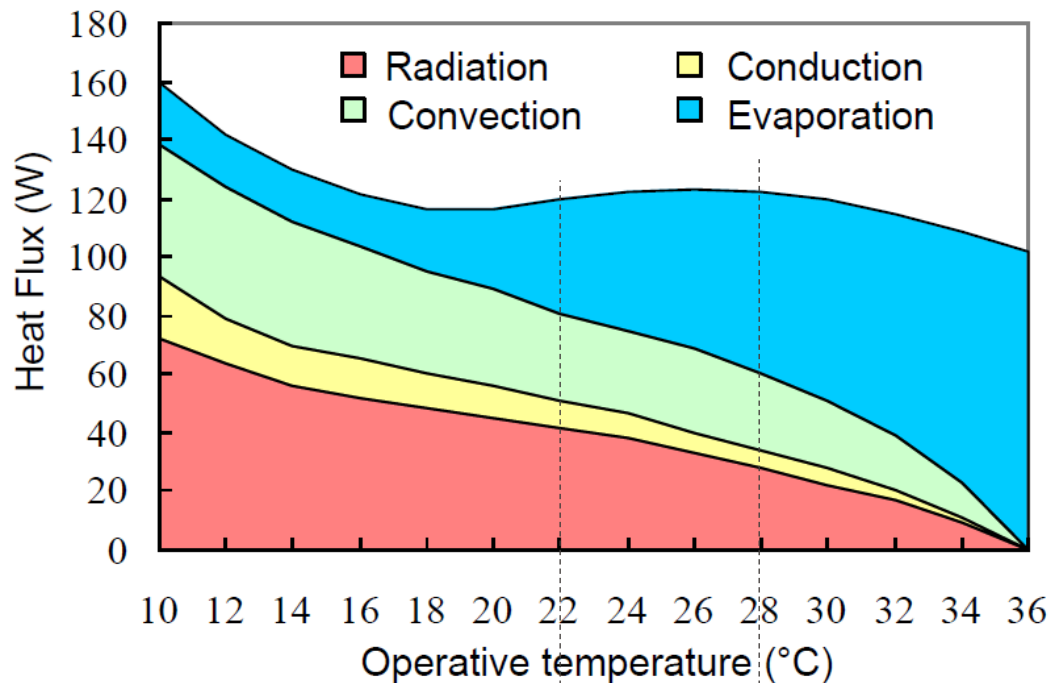
Clothing level

0.5 clo

t_{ab} 35.0 °C
 rh 12.2 %
 W_a 4.3 g w/kg da
 t_{wb} 16.6 °C
 t_{dp} 1.7 °C
 h 46.2 kJ/kg

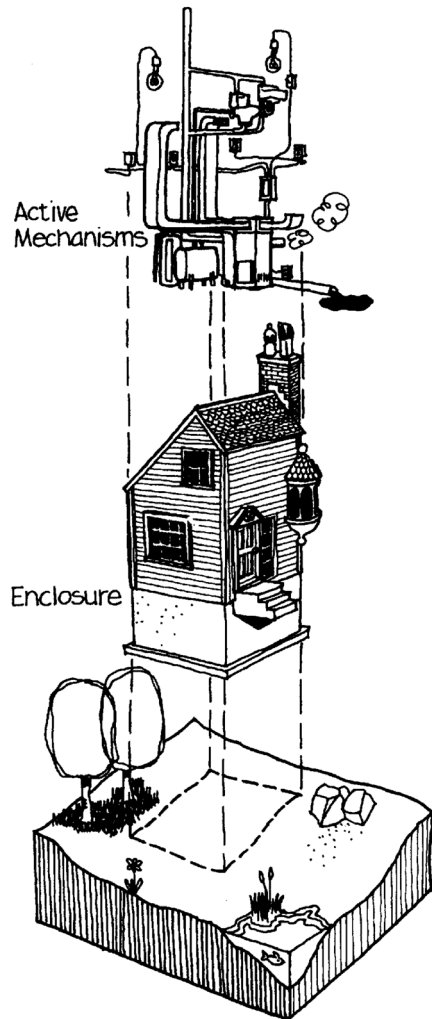


From Lect. 03:



←
Radiant heating is more efficient at low temperatures

→
Convective cooling is more efficient at high temperatures



Source: Edward Allen «How Buildings Work» (2005)

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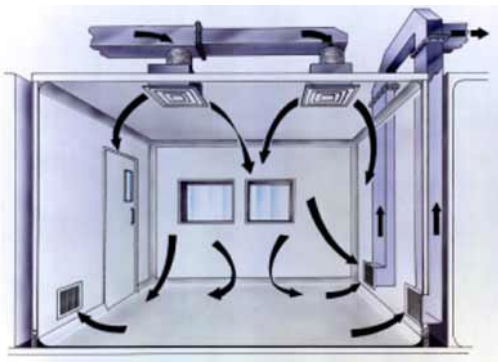
All-Air Systems

Radiant systems

centralized air conditioning

localized air conditioning

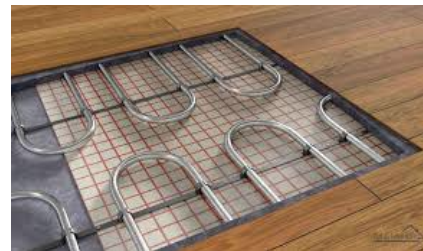
centralized water conditioning

heat extraction using: **air**heat extraction using: **refrigerant, water**heat extraction using: **water**

- Air supply through **grills** (no air deflection)

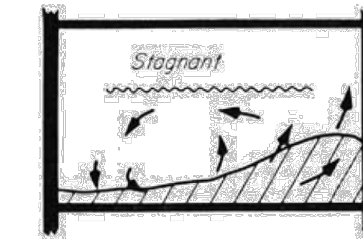
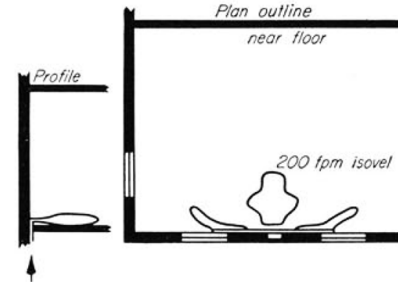
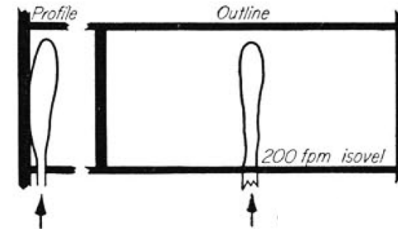
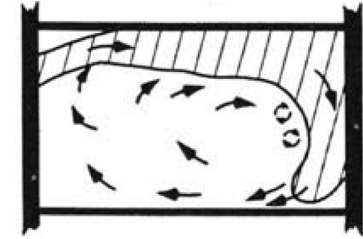
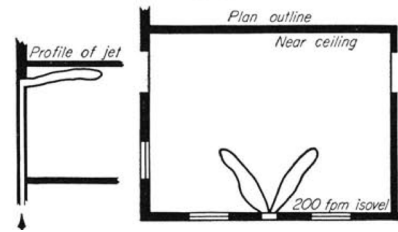


- Air supply through **diffusers** (diffuses air)

**Fancoils, indoor units** (air recirculation)**Chilled beams** (active and passive)
thermal conditioning +
fresh air supply (active only)**Radiant ceiling****Floor/Wall systems**

Distinct groups of air patterns:

- Air is discharged *at the ceiling level*
- Air is discharged *vertically* from floor grill (non-spreading floor outlet)
- Air spread *horizontally across the floor* (long baseboard units that spread air over a wide floor area)

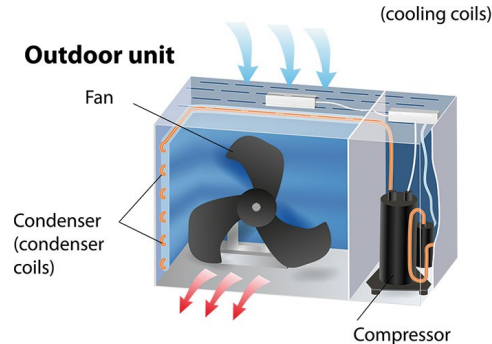
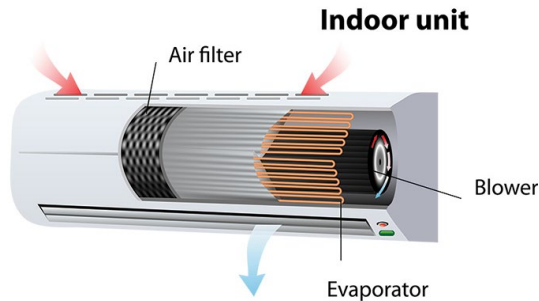


Split System

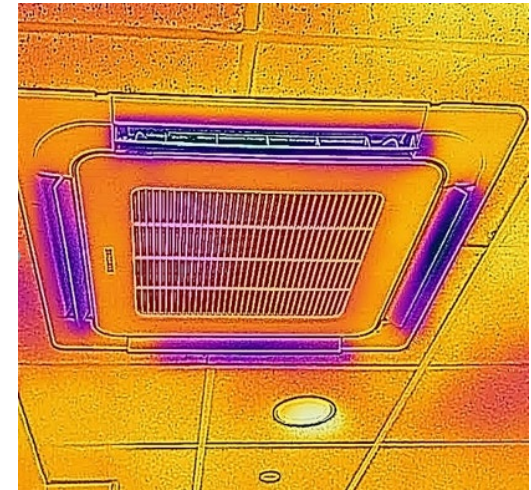
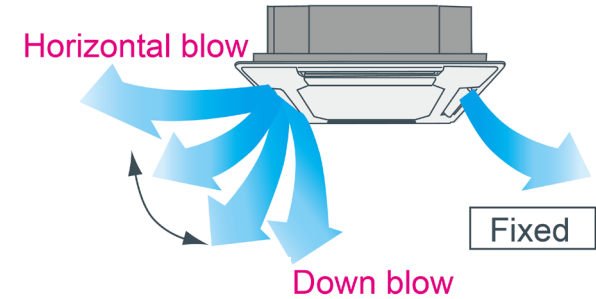
- A **split system** is the most common cooling equipment that serves a single zone (room)
- Indoor air cooling is based on the **vapor-compression cycle**.

Each split system has 2 units:

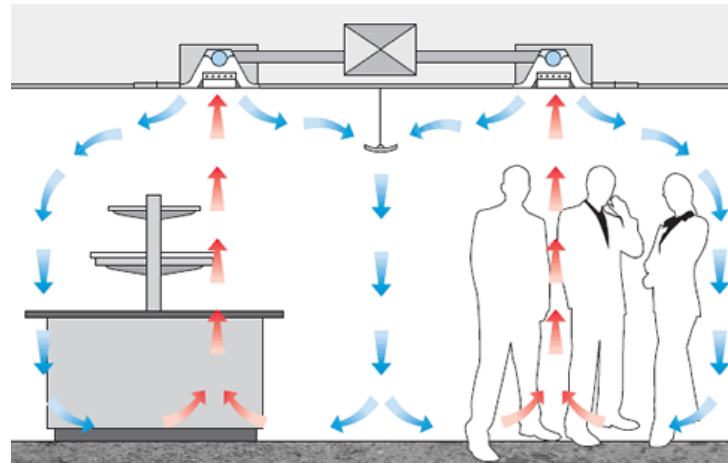
- **indoor unit** (an evaporator in a casing, with a fan blower)
- **outdoor unit** (a condenser with a fan, and a compressor in a casing)



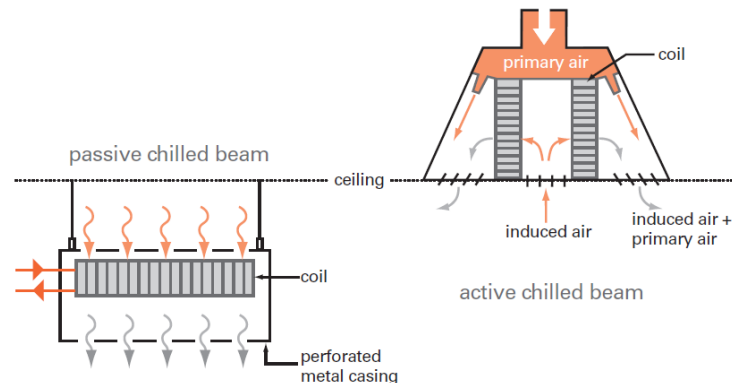
- Indoor terminal units that **remove heat from the space locally** *mainly* due to the **forced convection**
 - a room air is cooled by passing through a **fin-and-coil heat exchanger** that is filled with **circulating cold refrigerant**, a **fan** is used to force the airflow through the coil
- They **recirculate indoor air** (fresh air is provided by the separate ventilation system)
- A fancoil is **an evaporator** of the **vapor-compression cycle** that is used to “*generate*” cooling
- There can be multiple designs that can fit different space types, cold air is typically directed towards occupied zone:



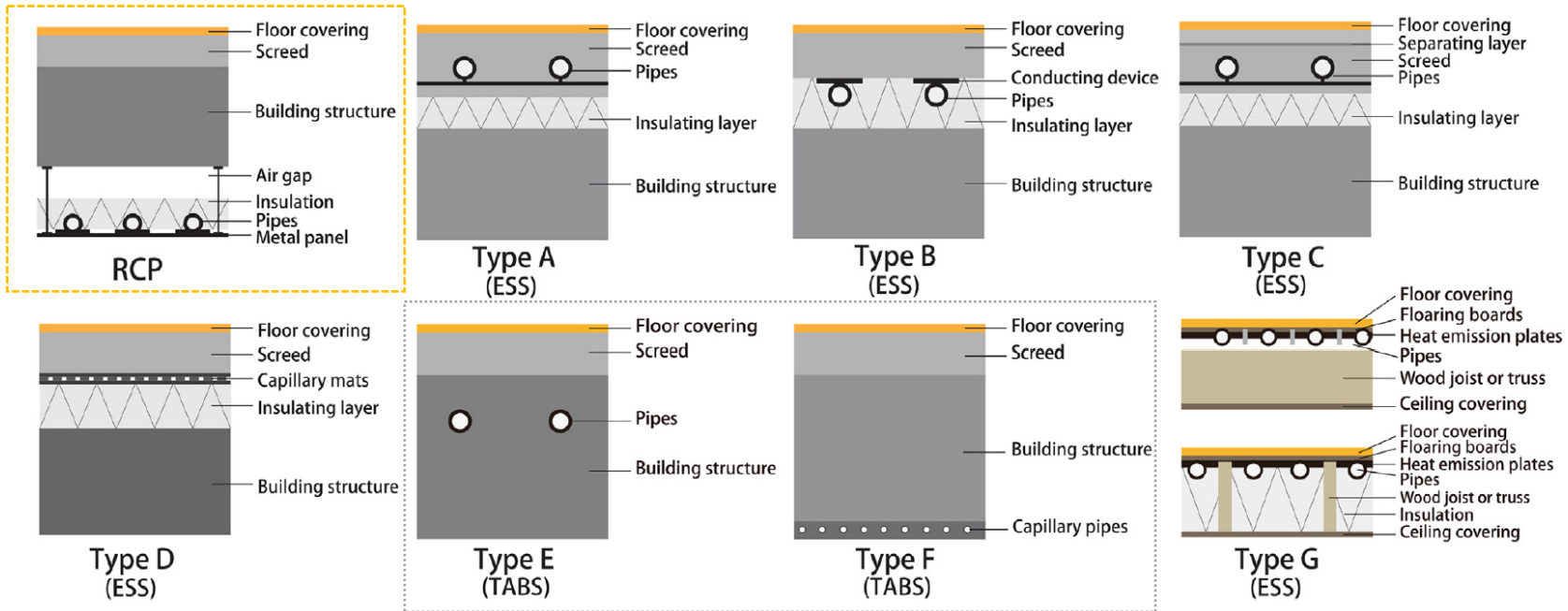
- Indoor terminal units that **remove heat from the space mainly due to convection**
- There are **2 types** of **chilled beams**:
 - **passive** (no fresh air supply, airflow circulation based on the buoyancy effect)
 - **active** (with integral air supply through nozzles)
- Both types have a *fin-and-tube type heat exchanger* which provides *air cooling* by removing heat using circulating cool water in pipes.
- The purpose of the **primary air system** for **active chilled beams** is to:
 1. Deliver *at least the required amount of outdoor air* to each space **for ventilation**
 2. Deliver **air** which is *dry enough* to **offset the space latent load** and **maintain the indoor dew point low enough to avoid condensation** on the chilled beams
 3. Deliver *enough air* to **induce sufficient room airflow** to *offset the space sensible cooling load*.

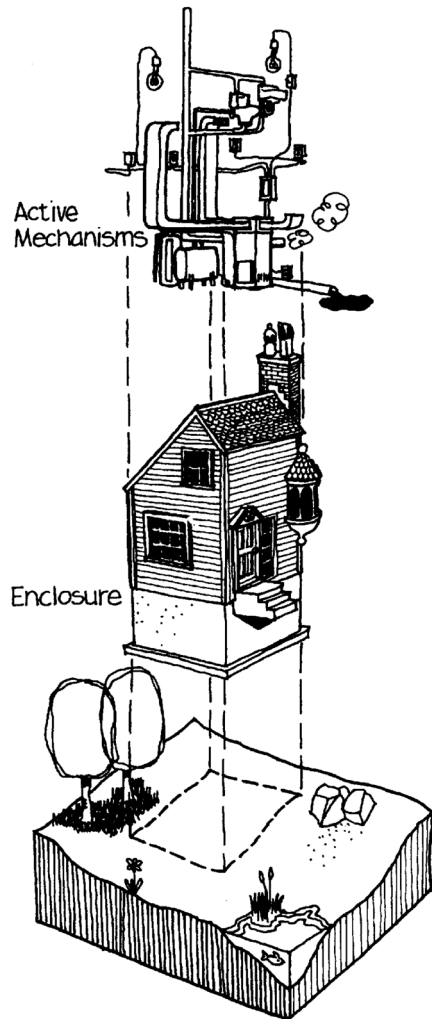


Source: TRANE, Engineering Newsletter, V38-4



- Three main categories of hydronic systems:
 - radiant ceiling panels (RCP),
 - embedded surface systems (ESS)
 - thermally activated building systems (TABS)





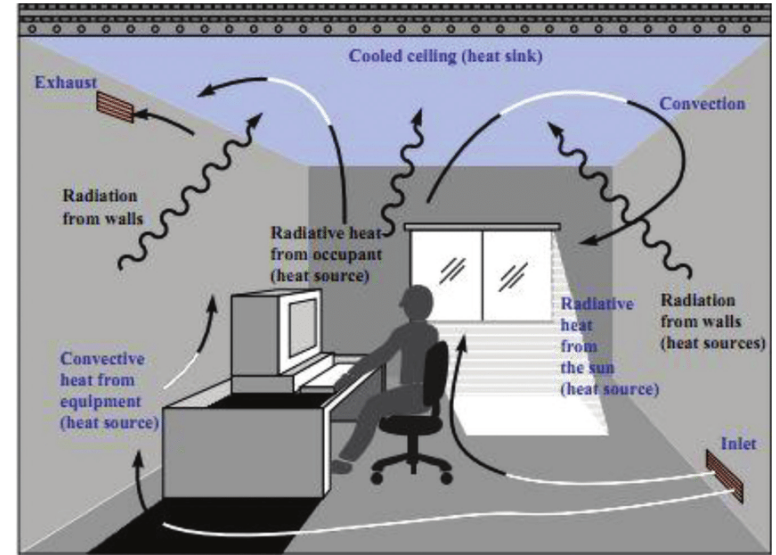
Source: Edward Allen «How Buildings Work» (2005)

CONTENT:

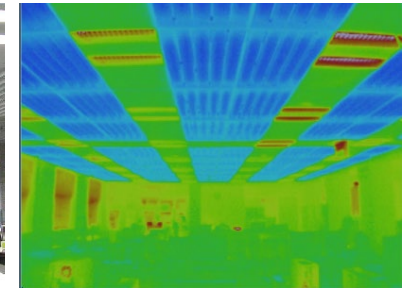
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All-Air Cooling vs. Radiant Cooling

- **Cooling** (rate of sensible and latent energy removal/extraction) required to maintain an indoor environment at a desired temperature and humidity
- **Heat extraction rate** – the rate at which heat is removed from a space by the emission system (terminal device)
 - For an all-air system: the enthalpy difference between airflow supplied to the space and airflow leaving the space
 - For a radiant system: the sum of convective and radiant heat transfer rates at the indoor face of internally cooled surface



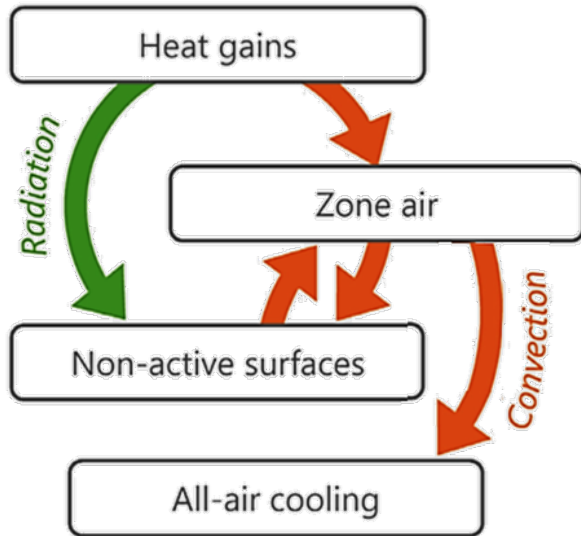
Source: Izzet Yüsek



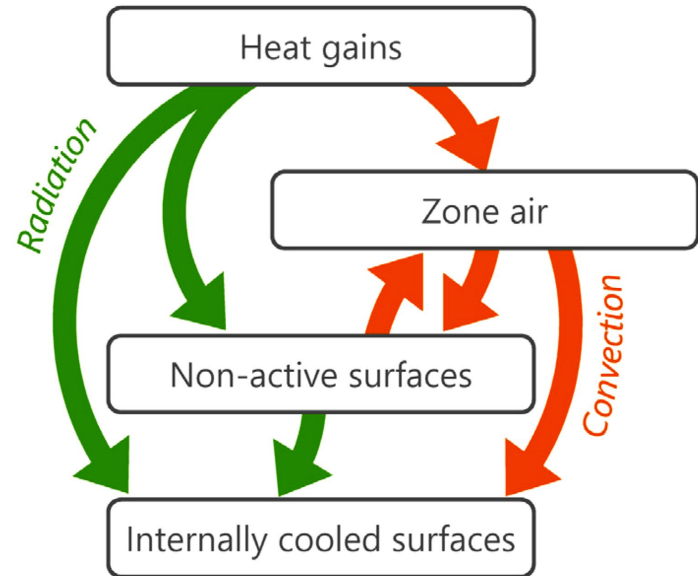
Source: SANKEN

Heat Transfer Pathways for Cooling

- In a space with **all-air cooling**, all radiant heat gains *are absorbed by non-active surfaces*.
- In a space with **radiant cooling**, a *portion of the radiant heat gains is absorbed by non-active surfaces and a portion is absorbed by the internally cooled surfaces*.

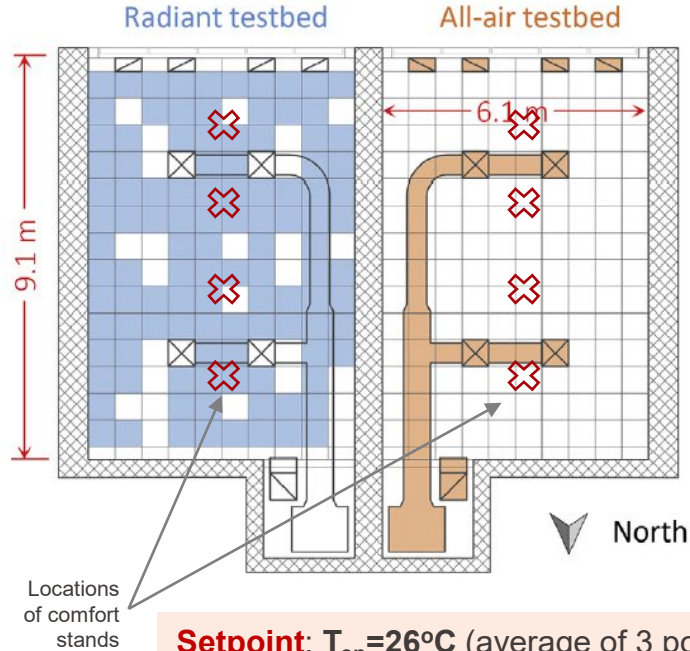


(a) all air cooling



(b) radiant surface cooling

All-Air Cooling vs. Radiant Ceiling Cooling



Setpoint: $T_{op} = 26^\circ\text{C}$ (average of 3 points at 0.6m height: 3.45 m, 5.3 m and 7.16 m from the south wall)

Equal comfort conditions


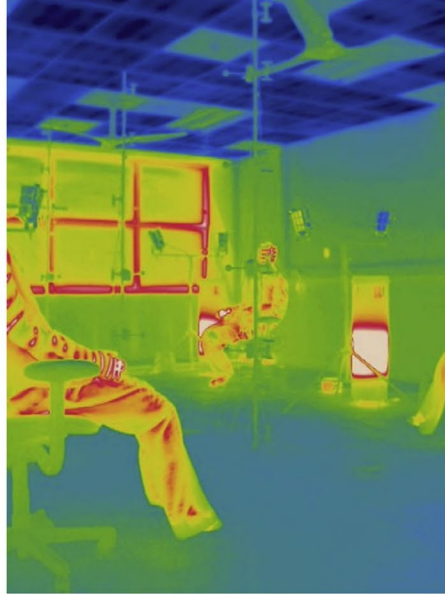
Radiant	Equal comfort conditions			All-air
	26 °C	Operative temperature	26 °C	
	≈ 26.5 °C	Air temperature	≈ 25.5 °C	
	≈ 25.5 °C	Mean radiant temp.	≈ 26.5 °C	

Source: Woolley et al. (2018) *Energy & Buildings* 176, pp 139–150

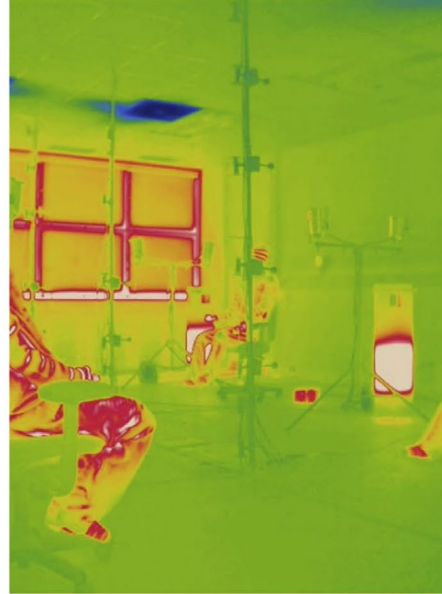
All-Air Cooling vs. Radiant Ceiling Cooling

Surface temperature (°C)

40
35
30
25
20
15

Infrared image of
radiant testbed



Infrared image of
all-air testbed



Visual image of testbeds

Equal comfort conditions

Radiant					All-air
	26 °C	Operative temperature	26 °C		
	≈ 26.5 °C	Air temperature	≈ 25.5 °C		
	≈ 25.5 °C	Mean radiant temp.	≈ 26.5 °C		

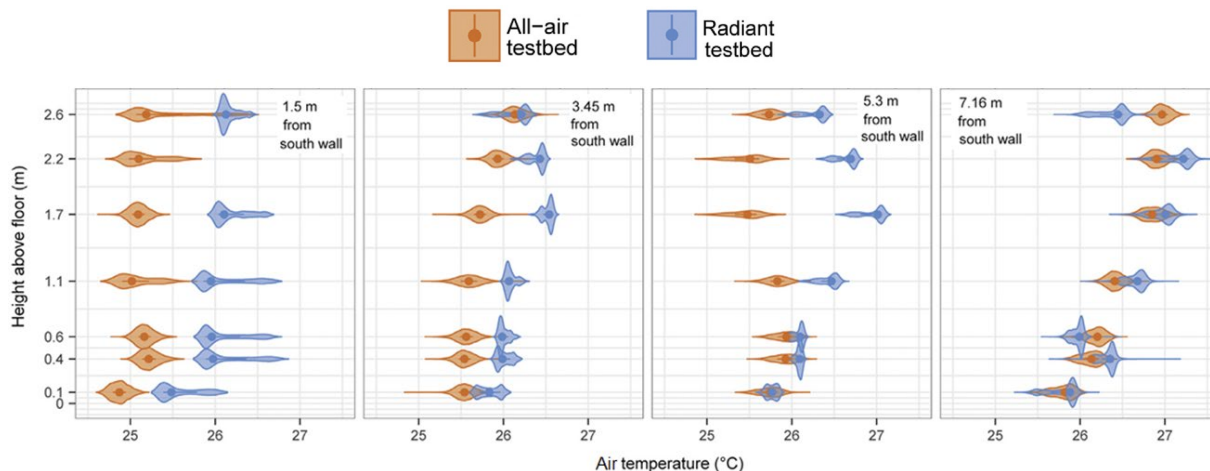
Environmental conditions

Outdoor air temp. ≈ 30 °C
Exterior surface temp. ≈ 60 °C

11:00-13:00@ 7 Sept. 2016

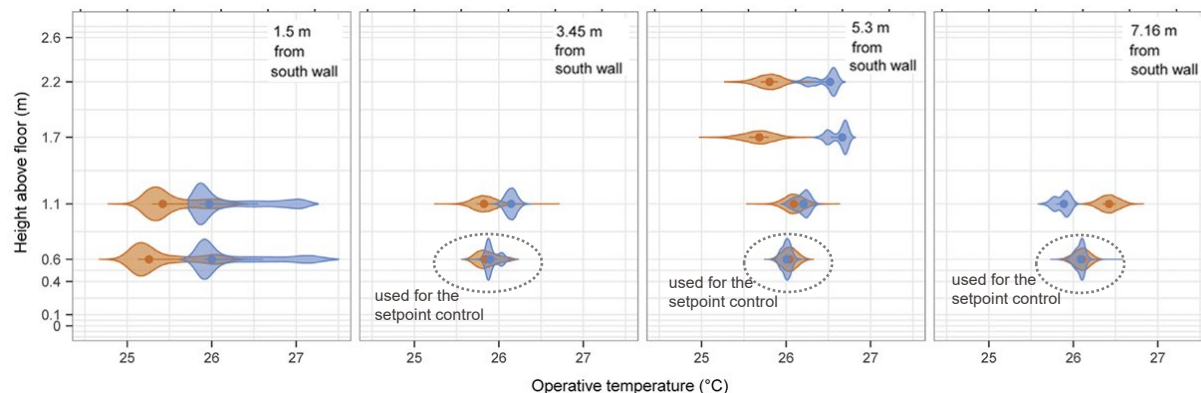
Air Temperature:

air temperature is **higher** in a **radiant cooled case** than in *all-air cooled case* at *equivalent comfort conditions* → radiant cooling could **reduce heat gain from ventilation** if supplied air is warmer.

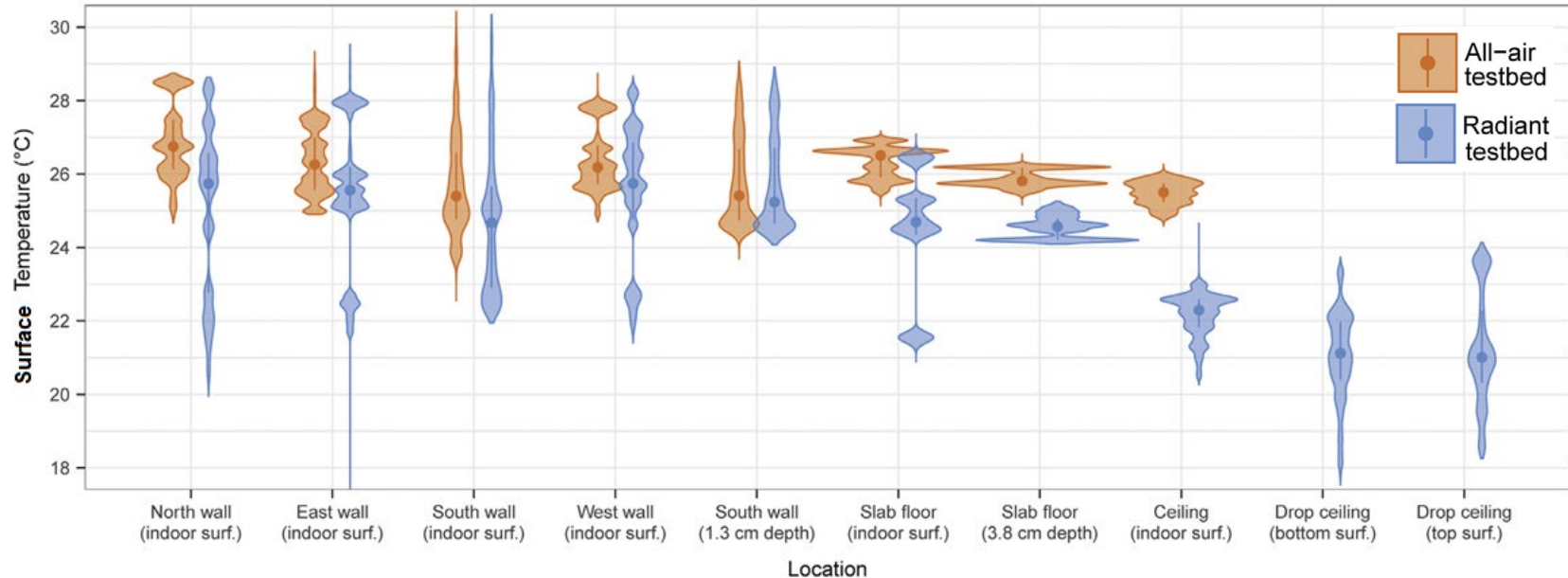


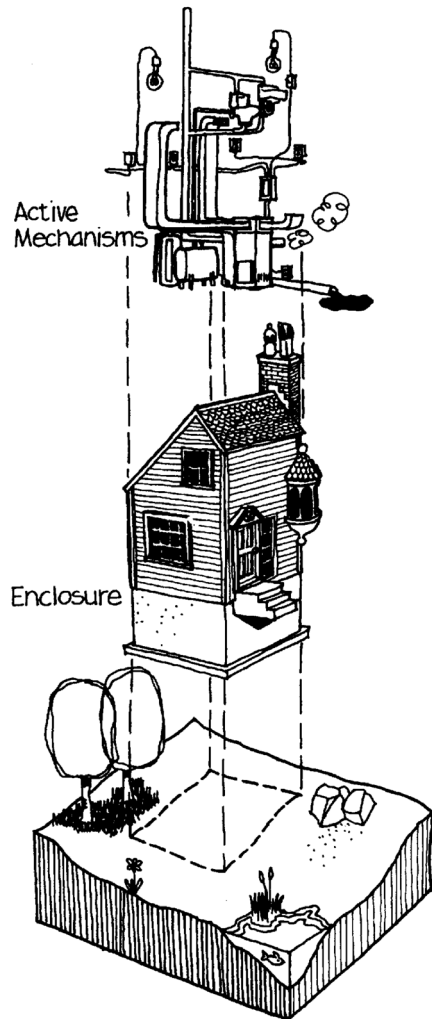
Operative Temperature:

variation throughout a room (in non-controlled locations) is a consequence of **non-uniform distribution of heat gains** and **distribution of cooling**



- **Surface Temperature:** All interior surfaces in a space with radiant cooling are cooler than in a similar space with all-air cooling → **net heat gain by heat transfer through the envelope can be higher**





Source: Edward Allen «How Buildings Work» (2005)

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EPFL Embedded Radiant Systems: **Rated Heat Flux**

- Surface convective heat transfer:

$$q''_{conv} = h_c \times (T_a - T_s)$$

- Longwave radiation heat transfer:

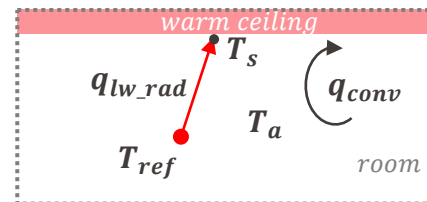
$$q''_{lw_surf} = h_{rad} \cdot (T_s - T_{ref})$$

- Total surface heat flux*:
(heat flow density, W/m²)

$$q'' = h_t \cdot |T_s - T_{ref}|$$

$T_s = T_{m,s}$ mean surface temperature

$T_{ref} = T_{op}$ operative temperature (per ISO 18555)



- Heat transfer coefficient** between radiant surface and room condition:

Combined convection and (longwave) radiation heat transfer coefficient h_t :

Heating:		Cooling	
Floor heating (FH)	11 W/m ² K	Floor cooling (FC)	7 W/m ² K
Wall heating (WH)	8 W/m ² K	Wall cooling (WC)	8 W/m ² K
Ceiling heating (CH)	6 W/m ² K	Ceiling cooling (CC)	11 W/m ² K

Radiant (longwave) heat transfer coefficient $h_{rad} = 5.5$ W/m²K
(less than 4% error for surface temperatures in the range of 15-30°C)

- Heat flow density (W/m²):**
at steady-state

○ **floor heating** / **ceiling cooling**:

$$q = 11 \cdot |T_{m,s} - T_{op}|$$

○ **wall heating** / **wall cooling**:

$$q = 8 \cdot |T_{m,s} - T_{op}|$$

○ **ceiling heating**:

$$q = 6 \cdot |T_{m,s} - T_{op}|$$

○ **floor cooling**:

$$q = 7 \cdot |T_{m,s} - T_{op}|$$

* Shortwave radiation heat transfer **is not considered** in the *standard approach* of defining total heat flow density which might become problematic **for floor cooling** with direct sunlight

What is the **comfort-related limiting heat flux** for **CEILING cooling** in the case of **indoor operative temperature 26°C** if **the surface temperature limit is 17°C**?

- A. 100 W/m²
- B. 42 W/m²
- C. 72 W/m²
- D. None above

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What is the **comfort-related limiting heat flux** for **FLOOR cooling** in the case of **indoor operative temperature 26°C** if **the surface temperature limit is 20°C**?

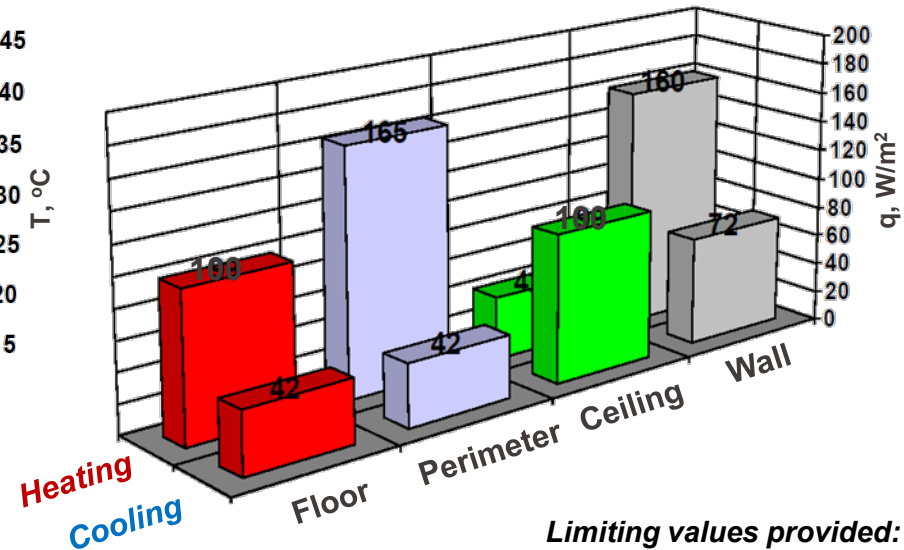
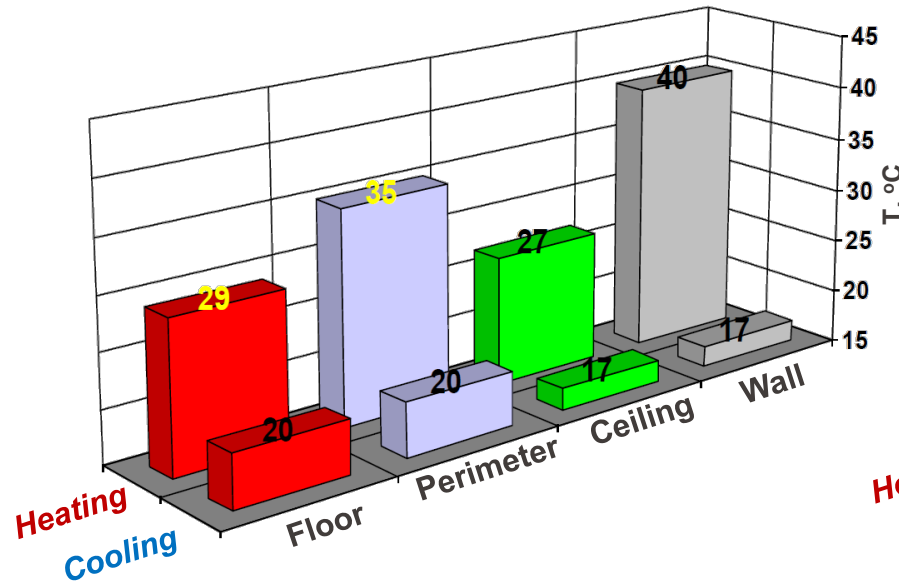
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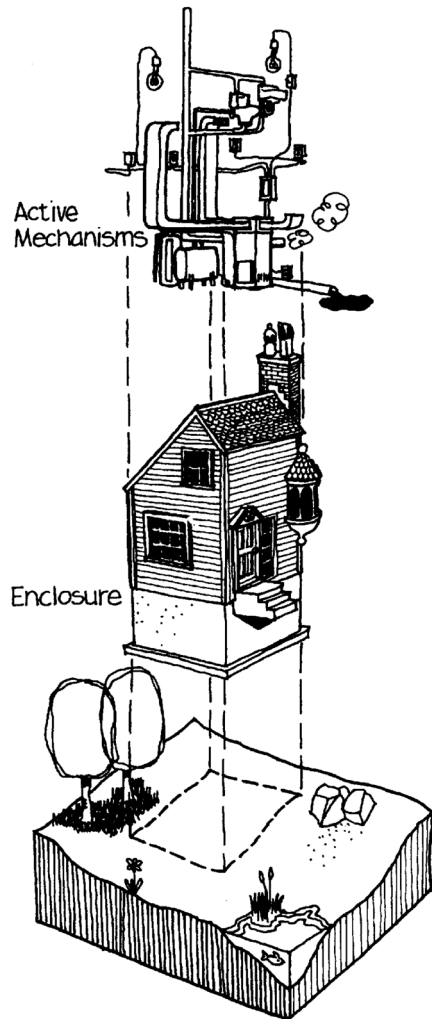
- Maximum heating and cooling capacity depends *on the position* of the radiant surface
- The heating/cooling capacity *is limited* by **relatively small convective heat transfer coefficient** between *the surface* and *air*, risk of condensation and concerns about surface temperature, radiant asymmetry, vertical air stratification and draft from comfort perspective



Limiting values provided:

- for **heating**: $T_{op}=20^{\circ}\text{C}$
- for **cooling**: $T_{op}=26^{\circ}\text{C}$

(for $V_{air} < 0.2 \text{ m/s}$)

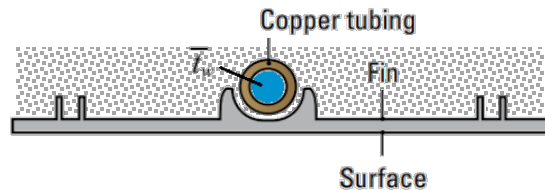


Source: Edward Allen «How Buildings Work» (2005)

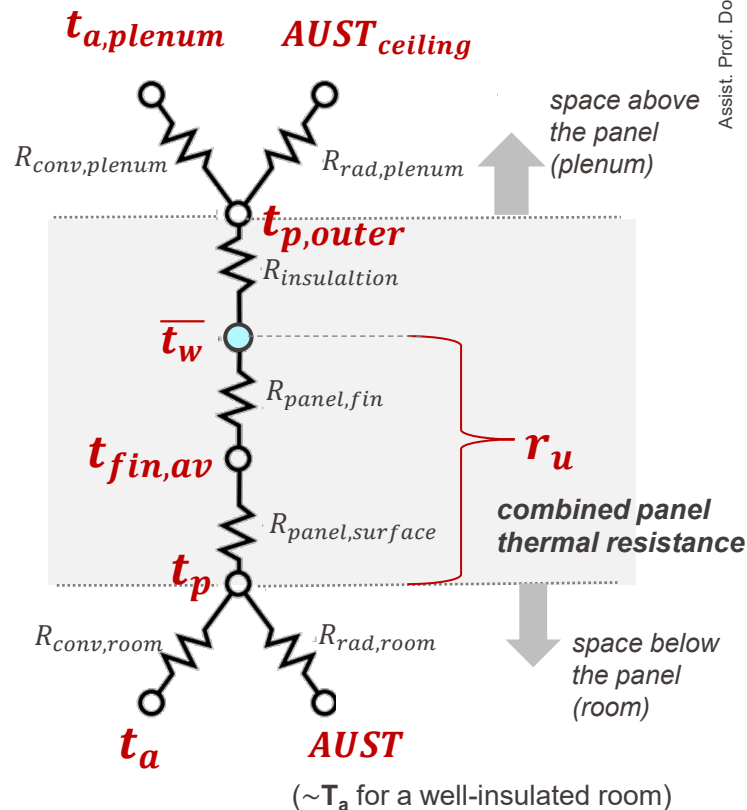
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- A radiant ceiling is a large heat exchanger suspended from a room's ceiling



- Thermal resistance in the panel to heat transfer from or to its surface reduces the performance of the system by affecting water temperature ($\overline{t_w}$)
- Average water temperature:** $\overline{t_w} = (T_{wi} + T_{wo})/2$
- Combined thermal resistance of the panel (r_u)** considers *architecture of the panel* (type of the bond between the piping and the panel material, spacing, piping, covering material presence, etc.)
 - When the *back side of the panels is insulated* using an insulation material, the **heat exchange toward the plenum could be ignored**



Thermal resistance of panels $(m^2 \cdot K)/W$:

$$r_u = r_t \cdot M + r_s \cdot M + r_p + r_c$$

- r_u - combined panel thermal resistance for a given spacing M of adjacent tubes
- r_t - thermal resistance of tube wall per unit tube spacing:

Circular pipes:

$$r_t = \frac{\ln(D_o/D_i)}{2\pi k_t}$$

Elliptical pipes:

$$r_t = \ln \frac{(a_o + b_o)/(a_i + b_i)}{2\pi k_t}$$

- r_s - thermal resistance between tube and panel body per unit spacing between adjacent tubes (see the tabulated values)
- r_p - thermal resistance of a panel body (see the table)
- r_c - thermal resistance of the active panel surface covering (i.e., pads, carpets, etc.), for thickness of x_c :

$$r_c = x_c/k_c$$

- k_t, k_p, k_c - thermal conductivity of tubing, panel, and covering materials, $W/(m \cdot K)$

Type of Panel	Thermal Resistance	
	r_p	r_s
<p>STEEL PIPE PAN EDGE HELD AGAINST PIPE BY SPRING CLIP ALUMINUM PAN</p>	$\frac{x_p}{k_p}$	0.32
<p>COPPER TUBE SECURED TO ALUMINUM SHEET</p>	$\frac{x_p}{k_p}$	0.38
<p>COPPER TUBE SECURED TO ALUMINUM EXTRUSION</p>	$\frac{x_p}{k_p}$	0.1
<p>METAL OR GYPSUM LATH TUBES</p>	$\frac{x_p - D_o/2}{k_p}$	≈ 0
<p>TUBES OR PIPES METAL LATH</p>	$\frac{x_p - D_o/2}{k_p}$	≤ 0.12

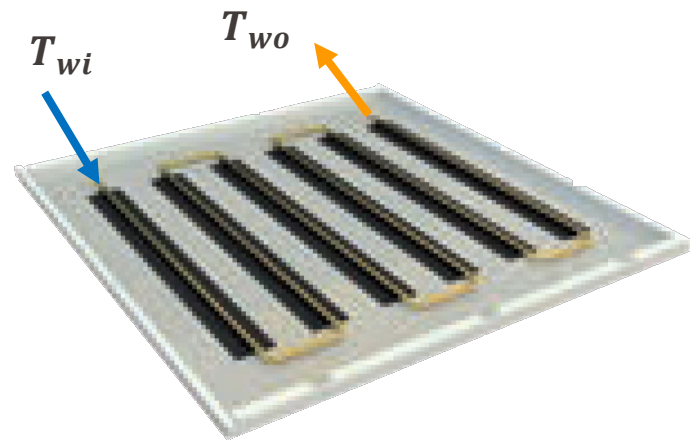
Source: ASHRAE Handbook 2020, HVAC Systems and Equipment, Chapter 6

Surface Cooling: Condensation Issues

- To prevent condensation on the room side of cooling panels, the panel water supply temperature (T_{wi}) should be maintained at least 1 K above the room design dew-point temperature (T_{dp}):

$$(T_{wi} - T_{dp}) > 1K$$

- This minimum difference is recommended to allow for the normal drift of temperature controls for the water and air systems, and also to provide a factor of safety for temporary increase in space humidity.
- Precooling and dehumidification of outdoor fresh air before it gets mixed up with the return air from the air-conditioned space, is recommended to prevent the surface condensation at panel surface



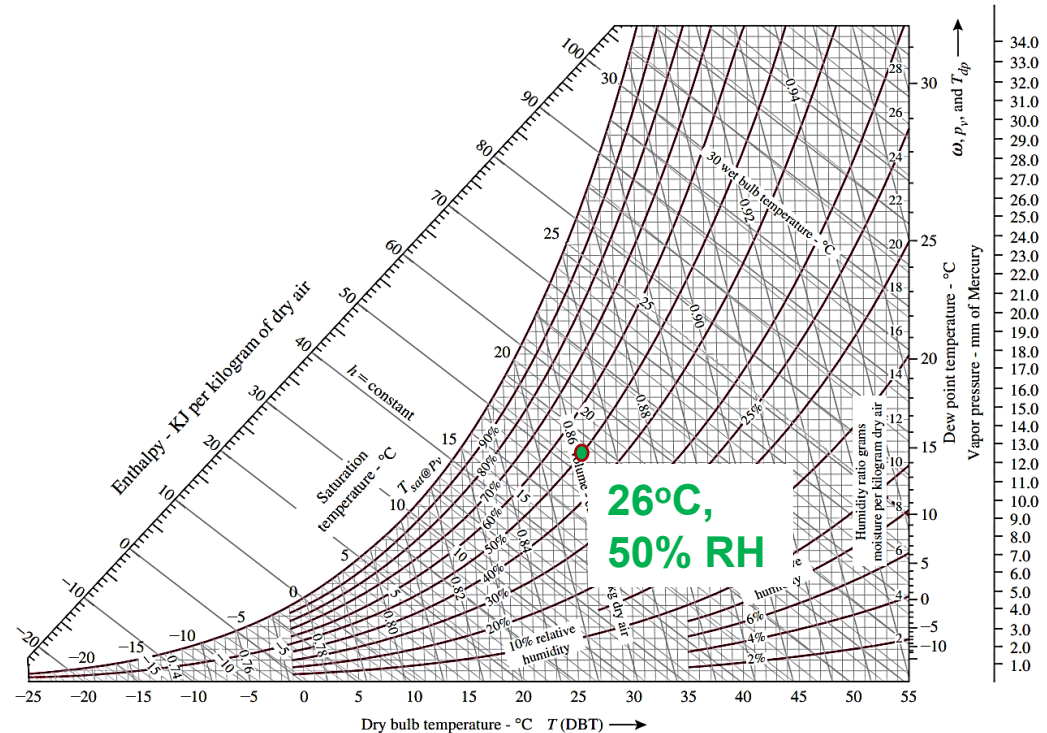
What is the dew point temperature when dry-bulb temperature is 26°C and relative humidity is 50%?

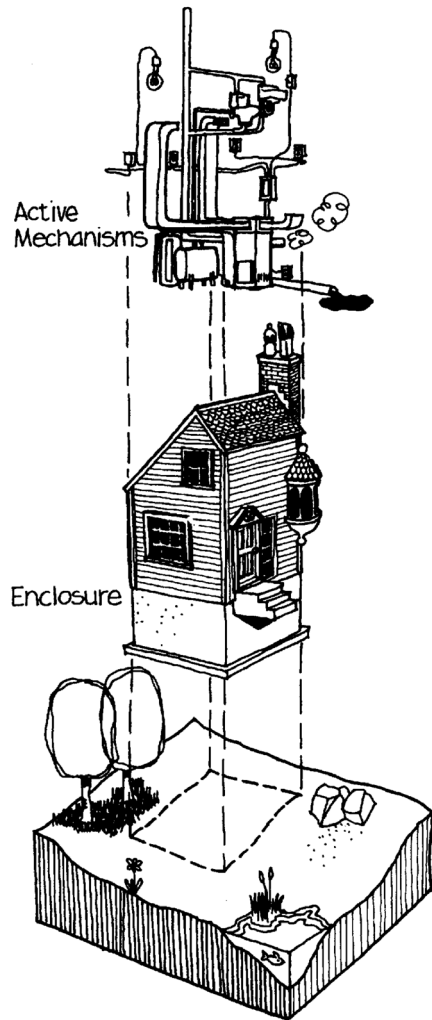
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- A. 13.8
- B. 14.8
- C. 15.0
- D. 26.0





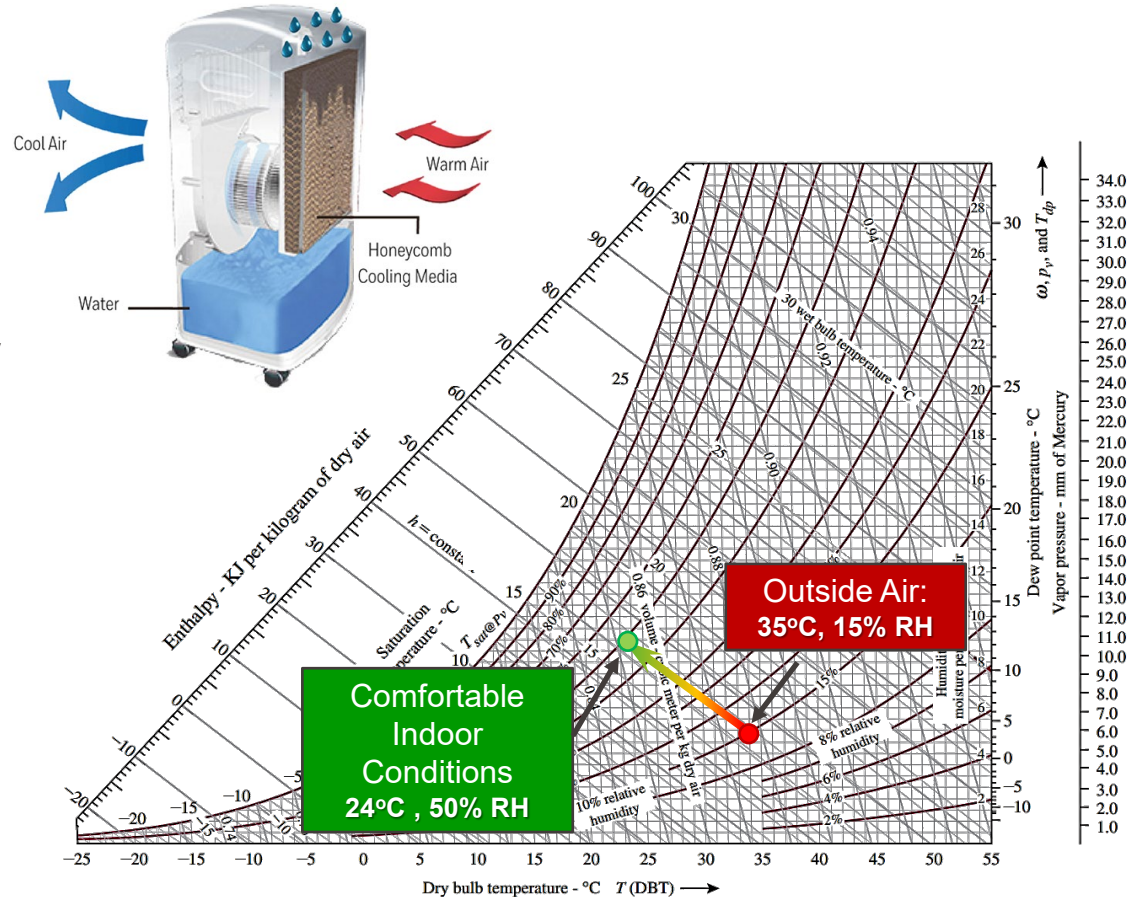
Source: Edward Allen «How Buildings Work» (2005)

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Evaporative Cooling

- *Simultaneous heat and mass transfer* process, or **adiabatic cooling**, and follows a **constant enthalpy line** on the psychrometric chart.
- The *cooling potential* for evaporative cooling is dependent on the **wet-bulb depression** (difference between the **dry-bulb** temperature and the **wet-bulb** temperature)
- Application is limited to **dry** locations



How much can **air temperature drop** if **evaporative cooling** is used for the following conditions:

- Ambient temperature: **28°C, 40% RH**
- Maximum desired RH is **60%**

- A.** 1°C
- B.** 2°C
- C.** 3°C
- D.** 4°C

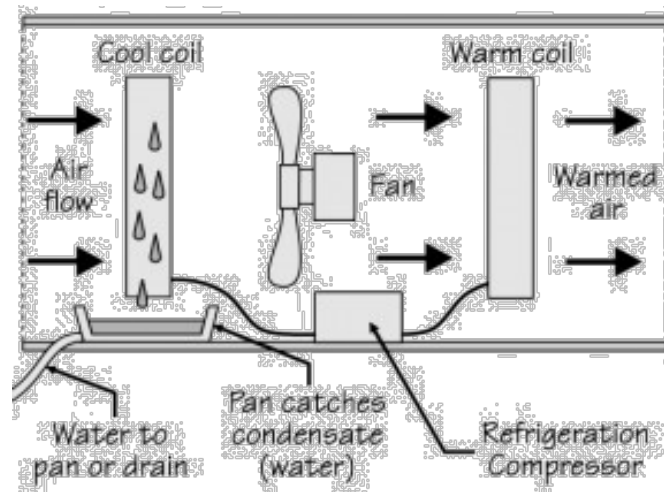
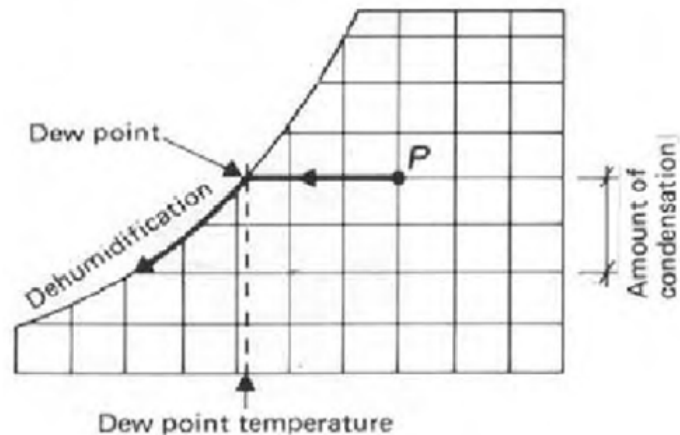
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EPFL Dehumidification

- **Dehumidification** – extraction of the water vapor that is contained in air
- The most common method of dehumidification: **sub-cooling of air** in **cooling coils** to **condense out moisture** using **chilled water** or **refrigerant**
- **Alternative strategies that** could be more energy efficient for certain climate zones and application types:
 - desiccant dehumidification (solid, liquid)
 - liquid desiccant dehumidification
 - dual wheel, wrapped around coil, *etc.*
- The *pre-cooling* and *dehumidification* of outdoor fresh air can be incorporated into general **air handling units (AHU)**.

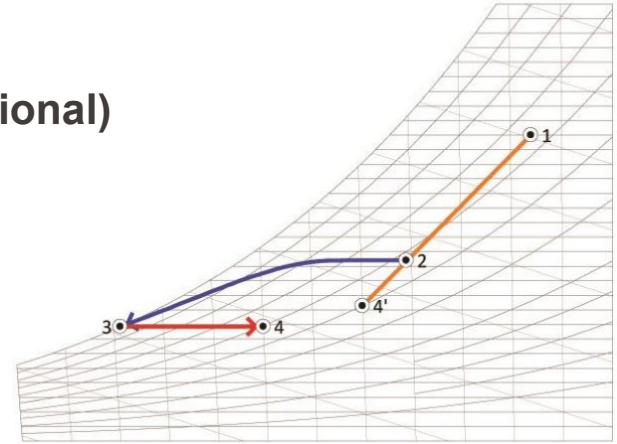
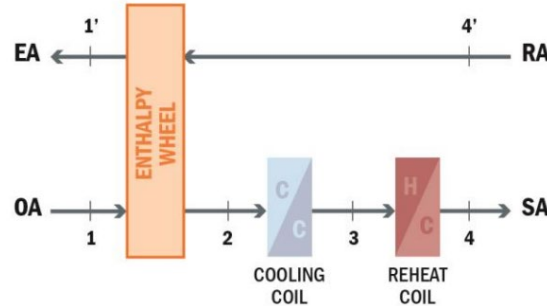


Chilled water cooling/ dehumidification (conventional)

1-2: heat recovery (energy exchange between the used air and entering fresh air)

2-3: air passes over the cooling coil to cool down to the required air humidity ratio

3-4: reheating to the required supply air temperature



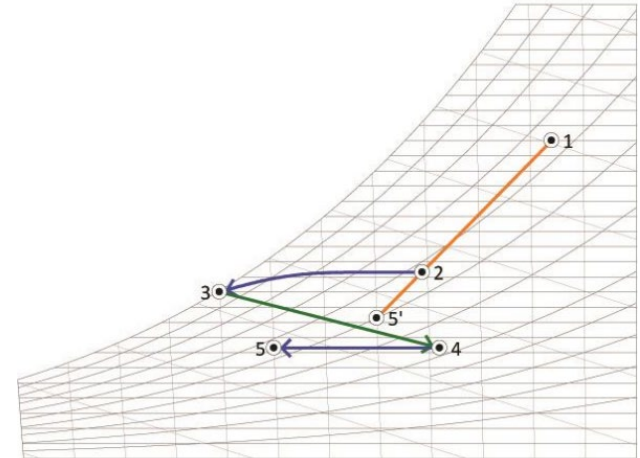
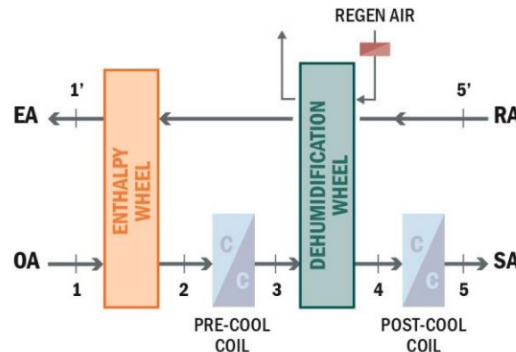
Solid desiccant dehumidification

1-2: heat recovery

2-3: air temperature reduction

3-4: isenthalpic dehumidification, the moisture content of the air reduces while air temperature increases

4-5: warm air passes over a post-cooling coil to cool it down to the required supply air temperature

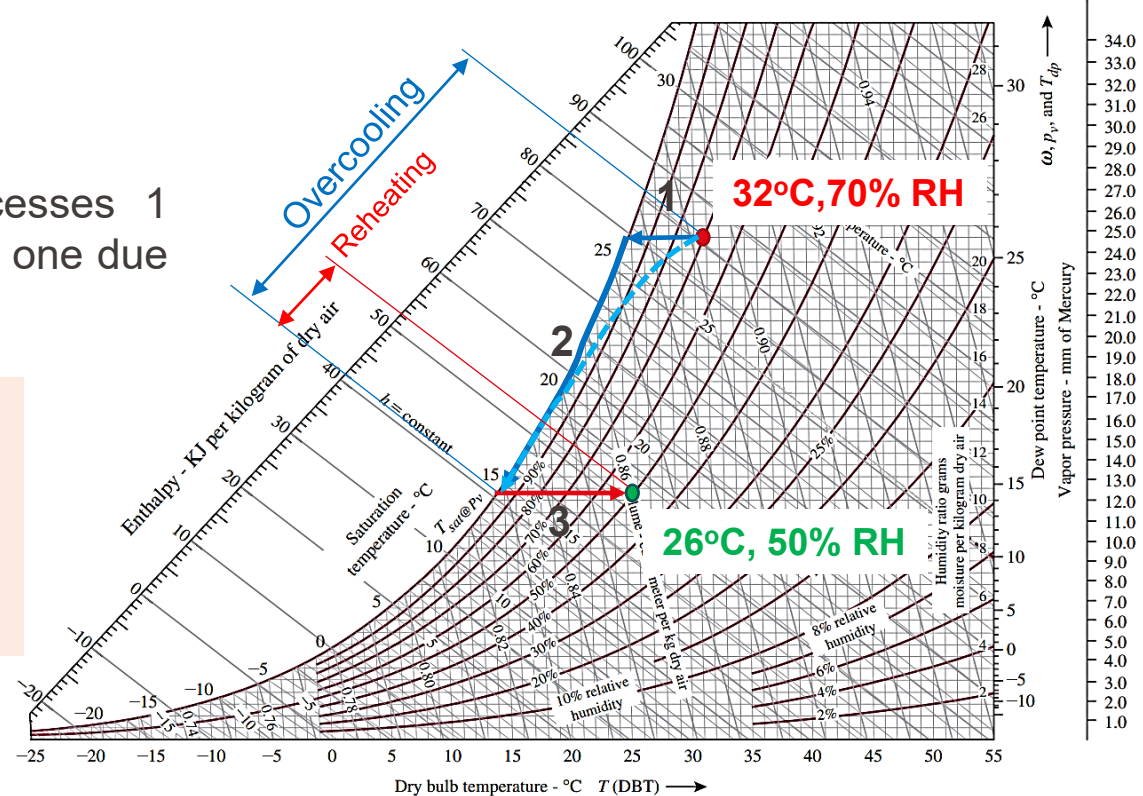


- Conventional dehumidification follows 3 processes:

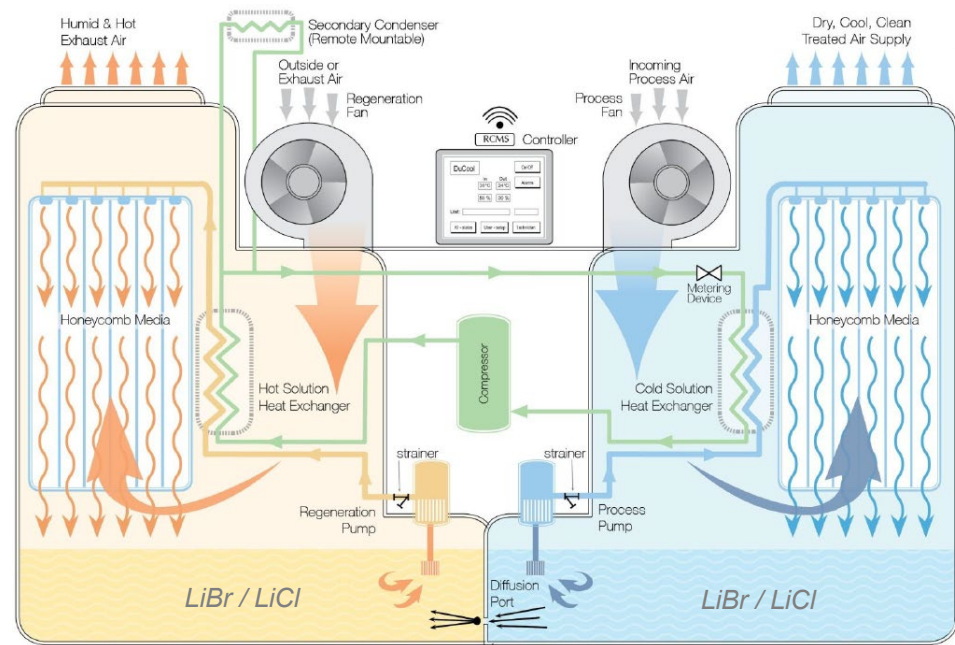
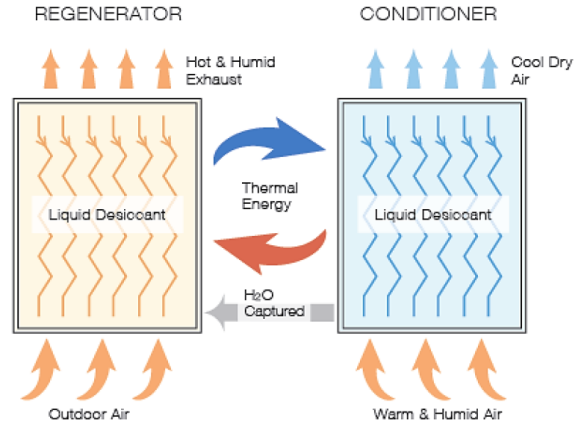
1. Sensible cooling
2. Latent cooling
3. Sensible heating

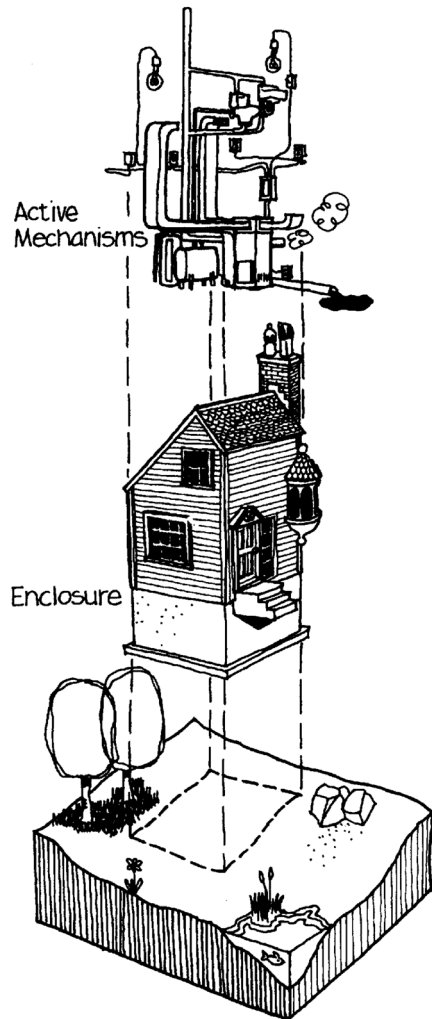
- In actual process, processes 1 and 2 deviate from ideal one due to the flow cooling.

- What is the *alternative more efficient process path* for the same given indoor and outdoor conditions?



■ Liquid desiccant dehumidification

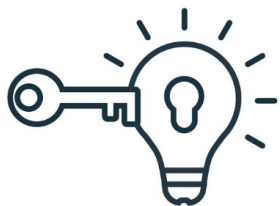




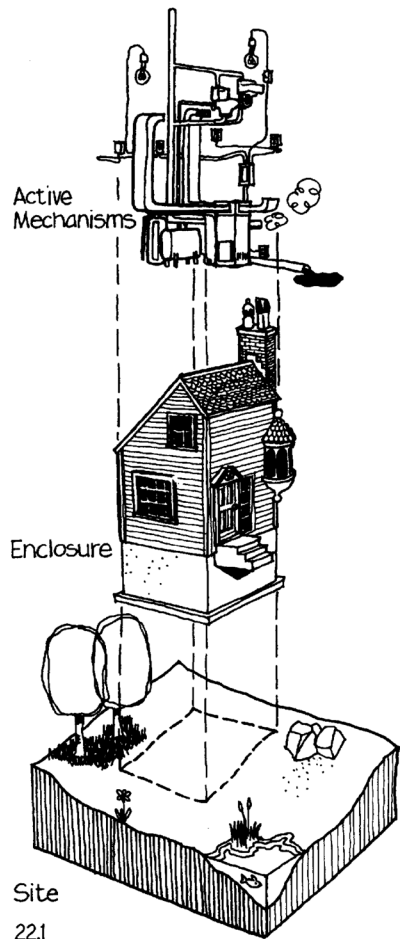
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- **Passive methods to keep buildings cool**
Preventing, modulating, and dissipating heat
- **Mechanical cooling emission systems categorization**
 - **All-Air Systems:** flow patterns depending on the supply direction
 - **Refrigerant-based systems:** split system, fancoils, chilled beams
 - **Hydronic Radiant Systems:** radiant ceiling panels
- **Radiant cooling:** heat transfer pathways, effect on the indoor environment, limiting heat flux, thermal resistance of radiant ceiling panels, condensation consideration
- **Evaporative cooling basics**
- **Dehumidification:** differences between conventional approach, solid vs. liquid desiccant dehumidification



Source: Edward Allen «How Buildings Work» (2005)

Thank you for your attention!

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<https://www.epfl.ch/labs/ice/>