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Session ID: **ENG445DK**



# **ENG-445**

# **Building Energetics**

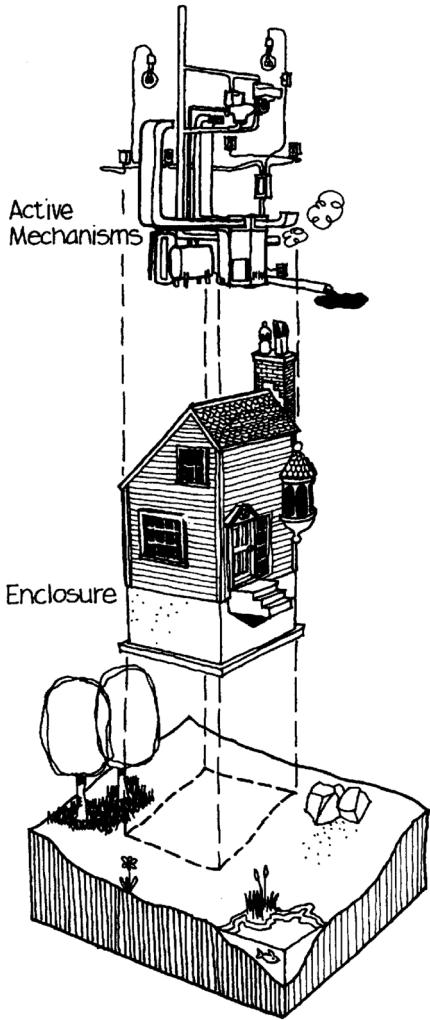
**Cooling in Buildings –  
Emission Systems and  
Thermal Comfort**

**Assist. Professor  
Dolaana Khovalyg**

28 November 2024

Week	Date	Topic	Timing	Teacher	Project (AS, MF)
9	07/11	Building envelope, thermal performance of building elements	45' x 2	DK	Tutorial building envelope
		Exercises			
10	14/11	Heating and cooling demand in buildings	45' x 2	DK	Free work
		Exercises			
11	21/11	Thermal systems for heating and their effect of human comfort	45' x 2	JY	Free work
		Exercises			
12	28/11	Thermal systems for cooling and their effect of human comfort	45' x 2	DK	Tutorial heating systems (emission systems)
		Exercises			

JY – Jaafar Younes, a postdoc from the ICE lab

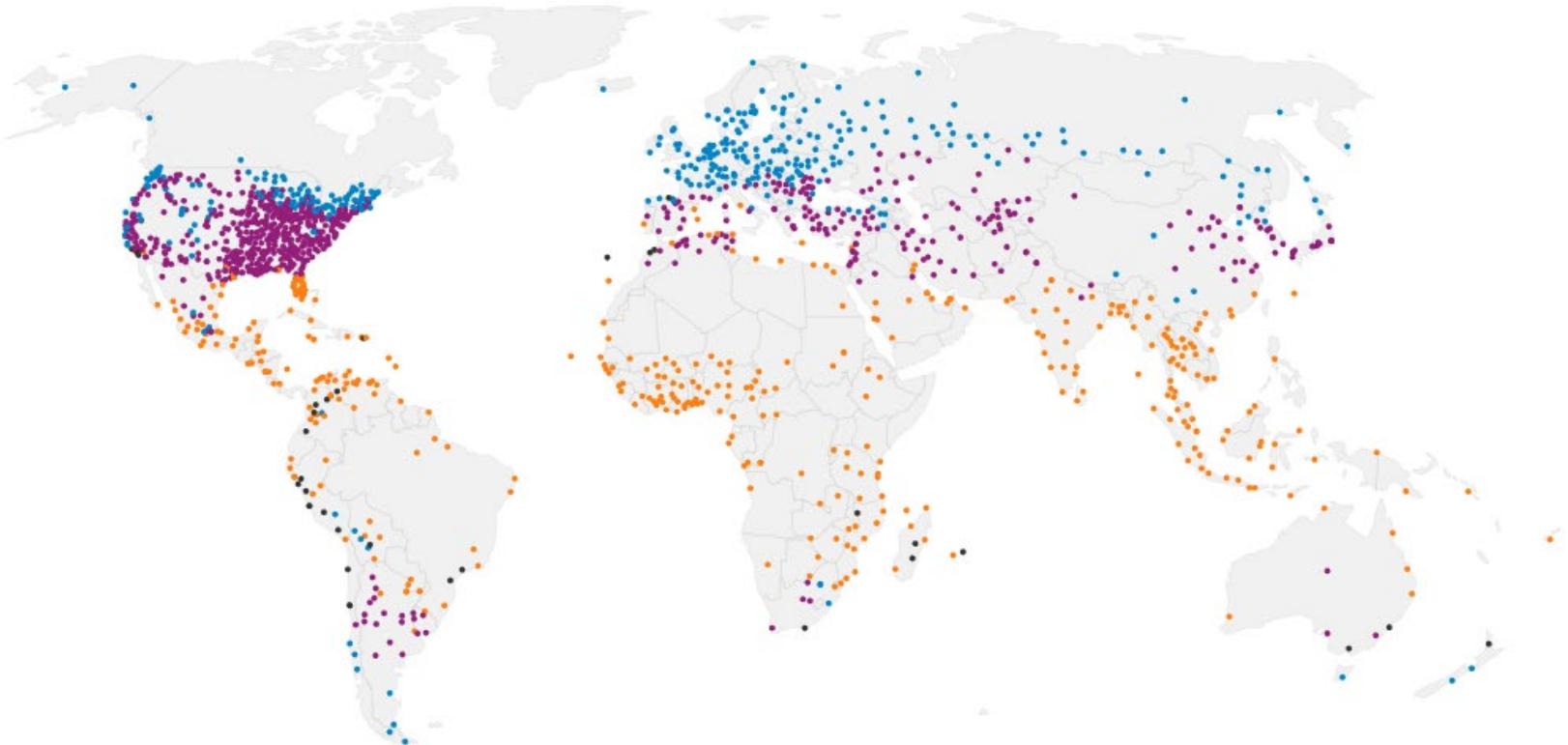


Source: Edward Allen «How Buildings Work» (2005)

# CONTENT:

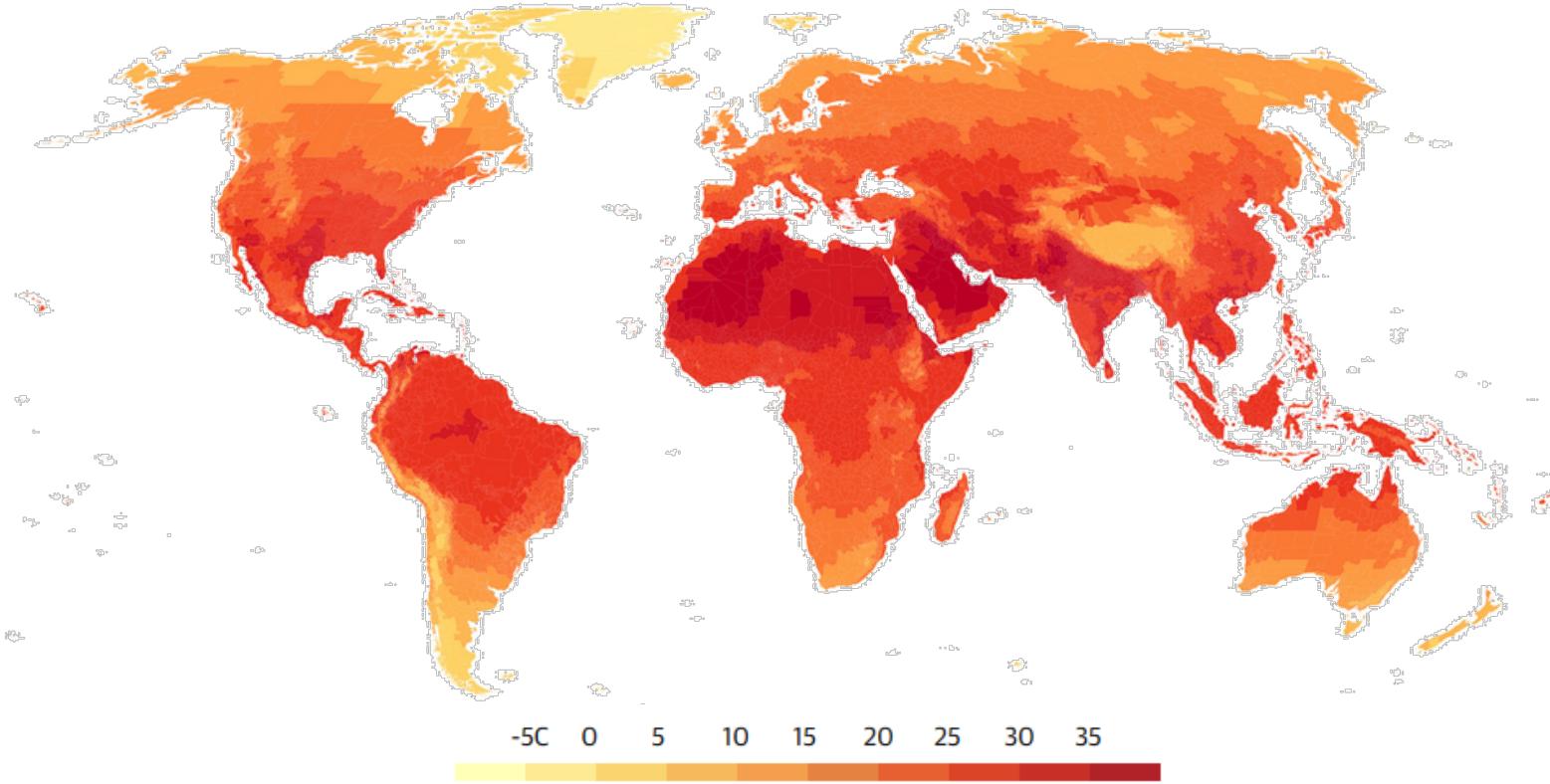
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  - **All-air, refrigerant-based, hydronic systems**
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    - Heat transfer overview
    - Limiting heat flux
    - Radiant ceiling panels
  - **Evaporative cooling**
  - **Dehumidification**
- **Summary**

# Indoor Thermal Conditioning: Overview



# Indoor Cooling: Increasing Needs

Average temperature between June and August, **2040-2059**



Source: <https://www.theguardian.com>

# Indoor Cooling: Increasing Needs



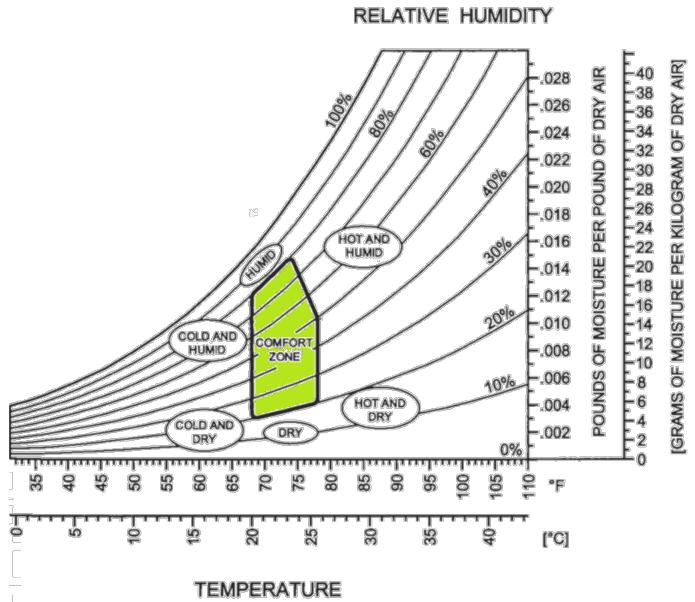
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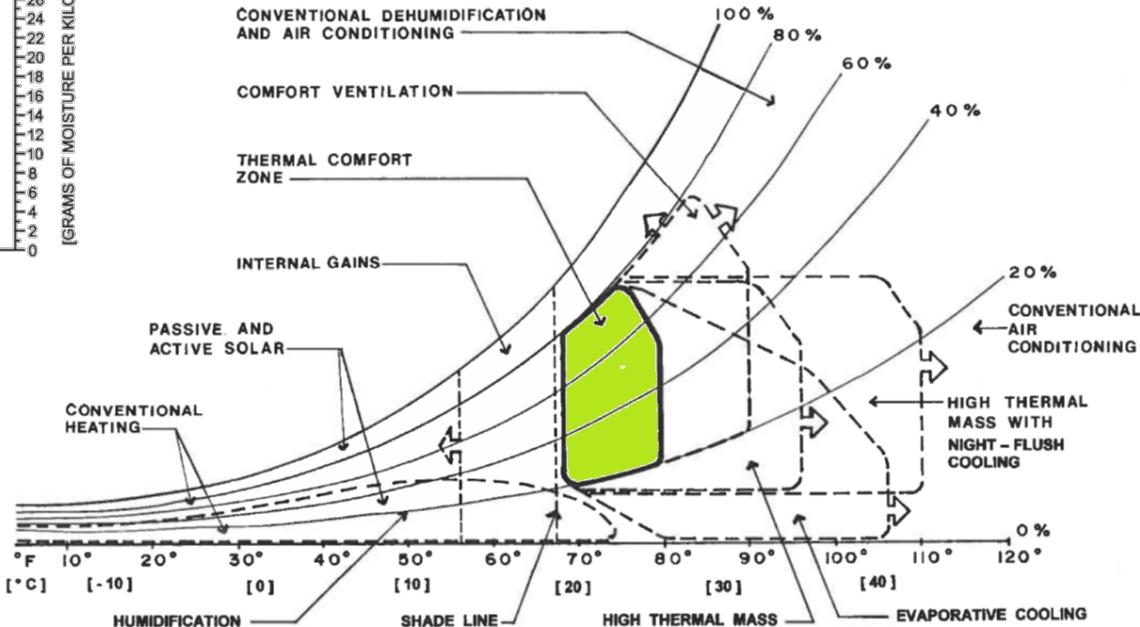
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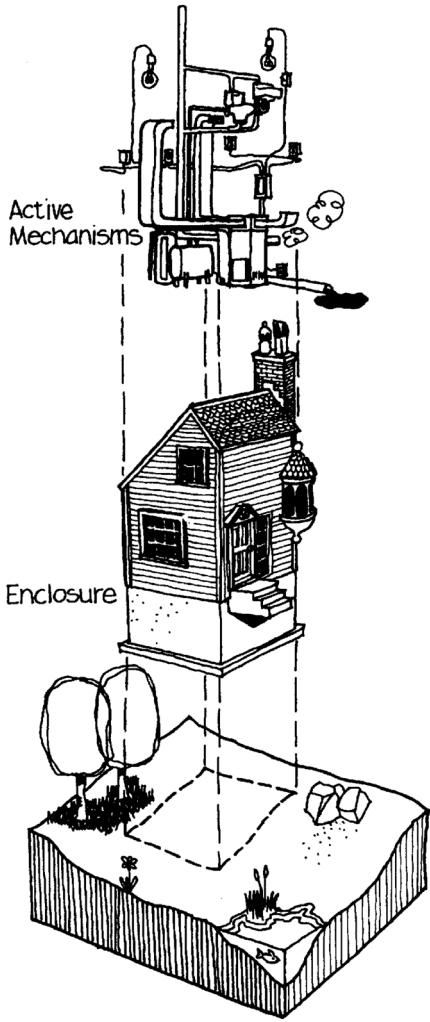


From Week 10



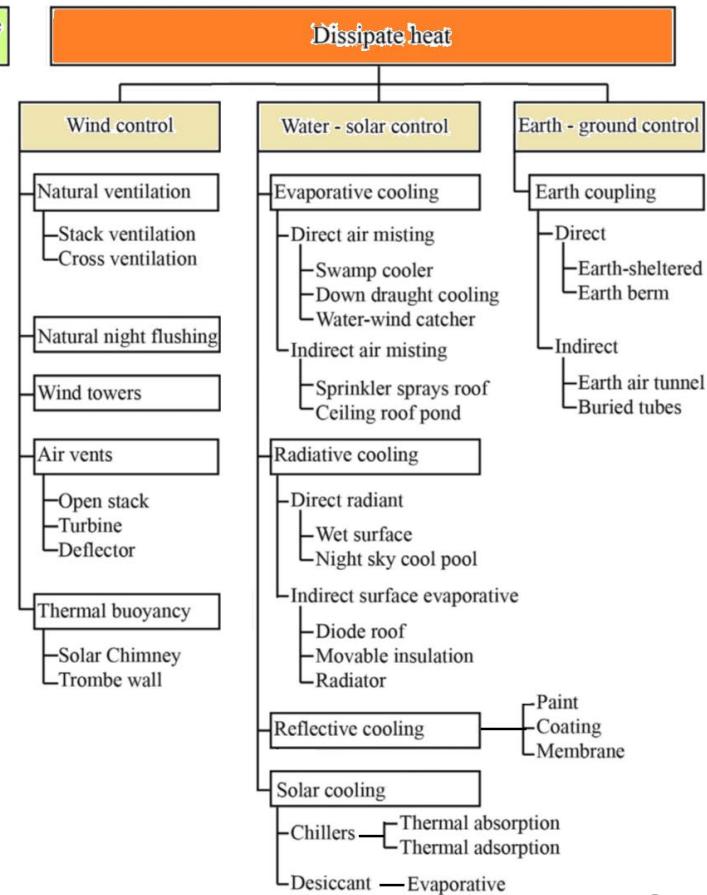
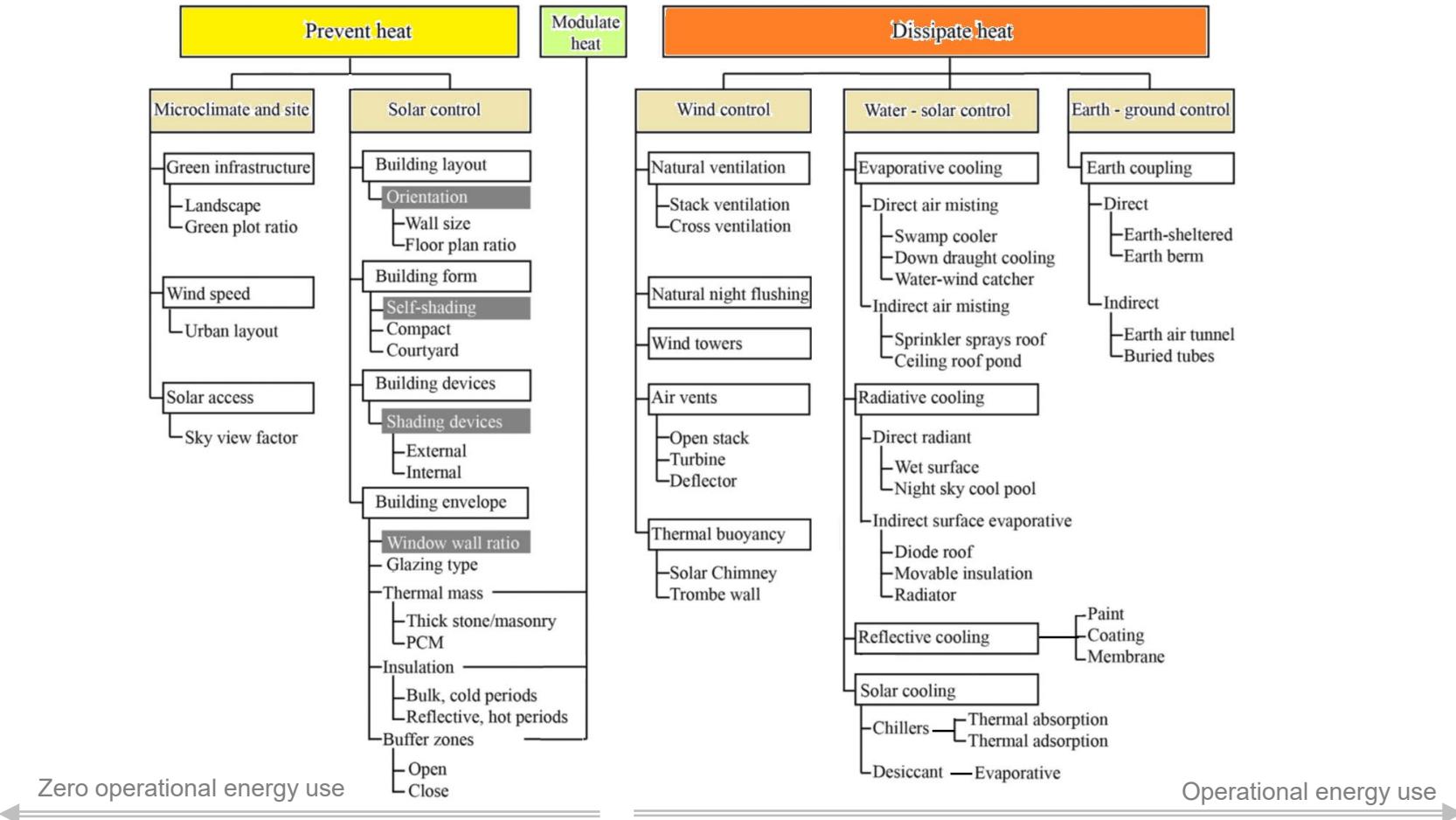
Climate analysis is necessary due to the **intrinsic relation between climate and energy demand for thermal comfort in buildings**





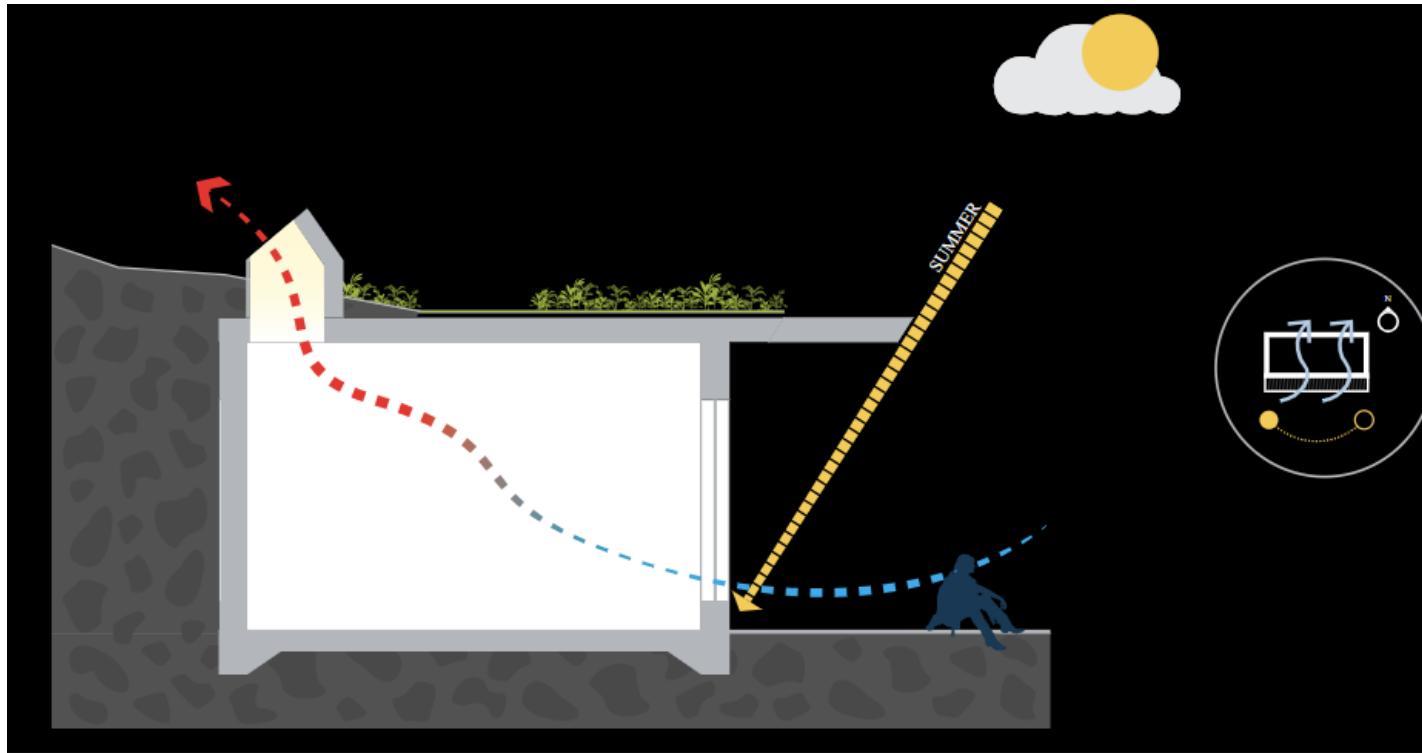
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EPFL Passive Cooling Strategies: Demonstration

<https://climatescout.arcadis.com/>



# EPFL Passive Cooling Strategies: Demonstration

<https://climatescout.arcadis.com/>



Form for Cooling



Cool Roof



Green Roof



East/West Shading



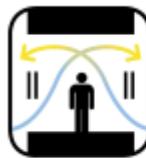
Solar Shading



Shading Devices



Stack Ventilation



Cross Ventilation



Night Vent Cooling



Building Facades



Earth Sheltering



Evaporative Cooling  
Towers

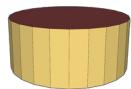
# What kind of building shape is preferable for warm locations?

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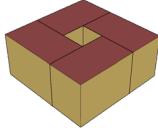
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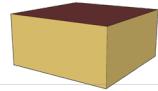
A.



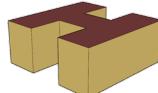
B.



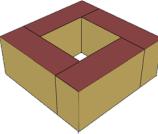
C.



D.



E.



# What are the conditions to keep buildings cool using the thermal mass?

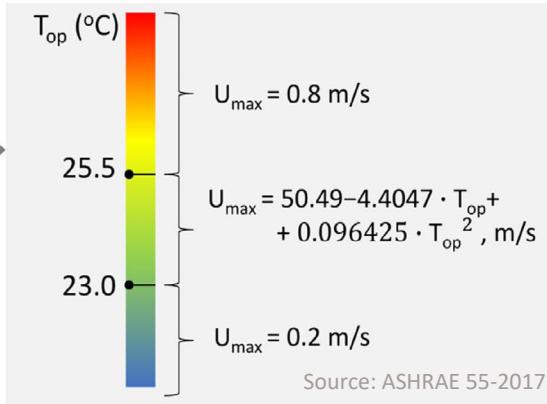
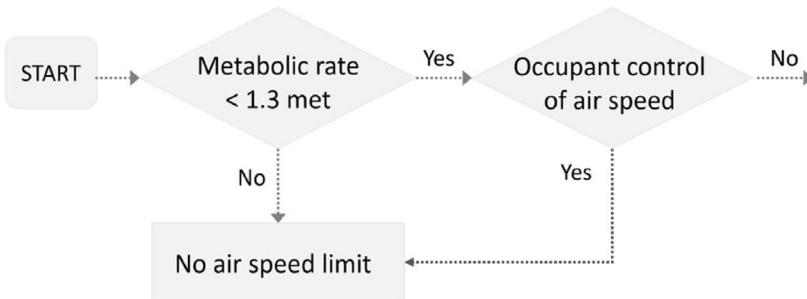
Discuss this question with your neighbors

- A. Sufficient diurnal outdoor temperature fluctuations
- B. Use concrete in building structure
- C. Insulate building envelope
- D. Avoid heat gains during the day
- E. Enable nighttime flashing
- F. All above

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1. Choose the building envelope configuration that **reduces** heat gains
2. Reduce solar heat gains:
  - by choosing **clear colors/reflective surfaces** for external opaque surfaces
  - by **shading** the building with vegetation
  - by planning **windows with reasonable sizes** according to their orientations
  - by planning efficient **solar protections** for fenestration
3. Reduce **internal heat gains** from indoor electric appliances
4. Precisely define **thermally conditioned zones** and position the **thermal boundary** accordingly
5. Ensure *enough indoor elements* with **high thermal inertia** within thermal boundary zone in order to **reduce instantaneous overheating** the space
6. Remove the excess *heat* from the building using **passive means** (e.g., deploy **night-time cooling using natural ventilation** by planning openings)
7. Introduce simple cooling systems (e.g., **fans**, **evaporative cooling**)
8. Consider **active cooling only to deal with overheated periods** that cannot be reasonably avoided after applying steps (1)-(7)



Source: reviewsofairpurifiers.com



Downward cool air flow

**Summer**  
anti-clock wise rotation

Upward hot air flow

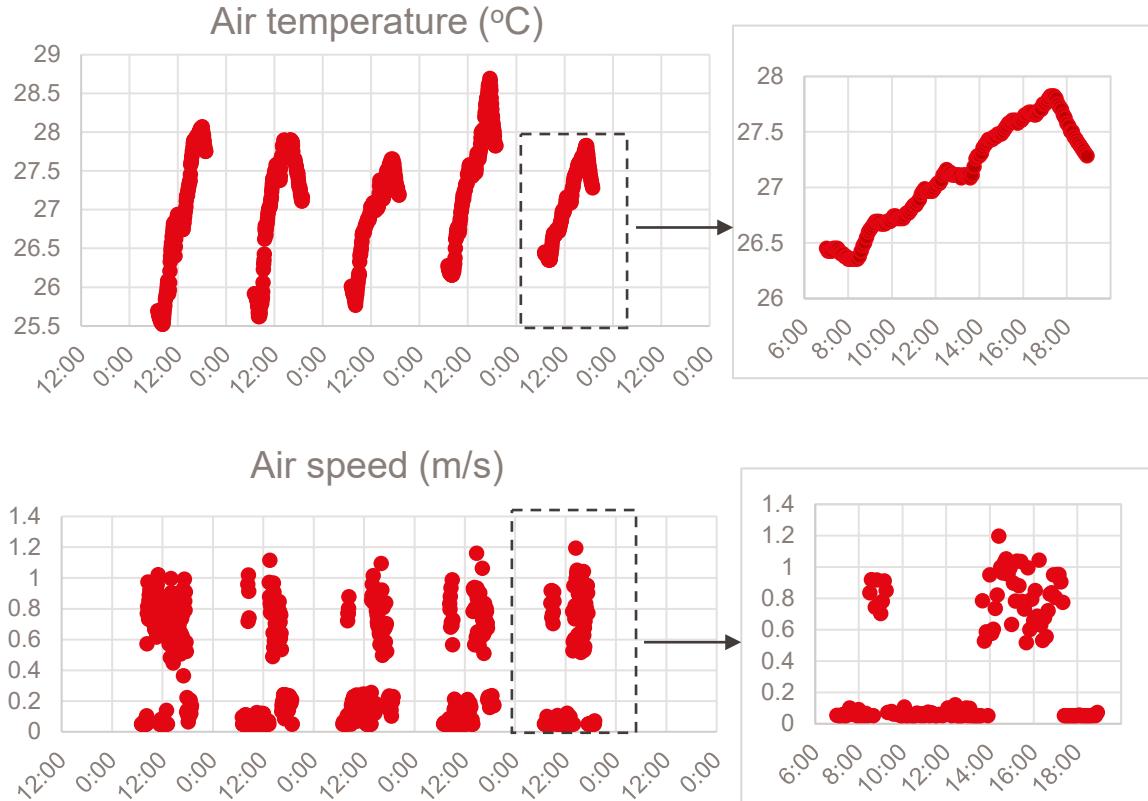


Upward cool air flow

**Winter**  
clock wise rotation

Downward hot air flow

- Example of **elevated air speed** created by **a person** to improve the comfort
- Office monitoring in Geneva, summer 2020 (**eCOMBINE** project)
- **2 weeks** of measurements
- **1/30** person was using a desk fan
- Maximum air speed was between **0.6-1.2 m/s**



- Fans elevate air flow around the human and increase convective heat removal from the skin
- Increased convection helps also to increase latent heat removal (via evaporation)
- Sensation scale correction up to 1 point (from  $PMV = +0.43$  to  $PVM = -0.48$ )

Operative temperature

28   °C

Air speed

0.2   m/s

Relative humidity

50   %

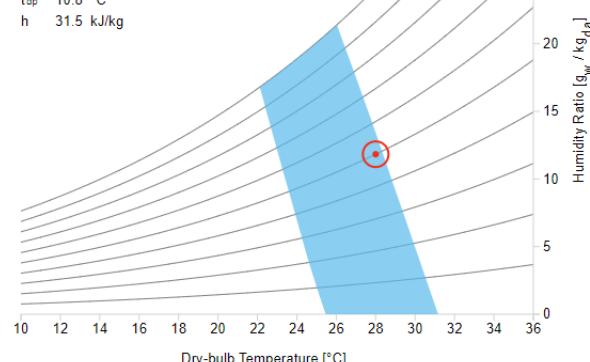
Metabolic rate

1   met

Clothing level

0.5   clo

$t_{db}$  11.0 °C  
 $rh$  99.8 %  
 $W_a$  8.1 g  $w/kg$   $da$   
 $t_{wb}$  10.9 °C  
 $t_{dp}$  10.8 °C  
 $h$  31.5 kJ/kg



Operative temperature

28   °C

Air speed

1   m/s

Relative humidity

50   %

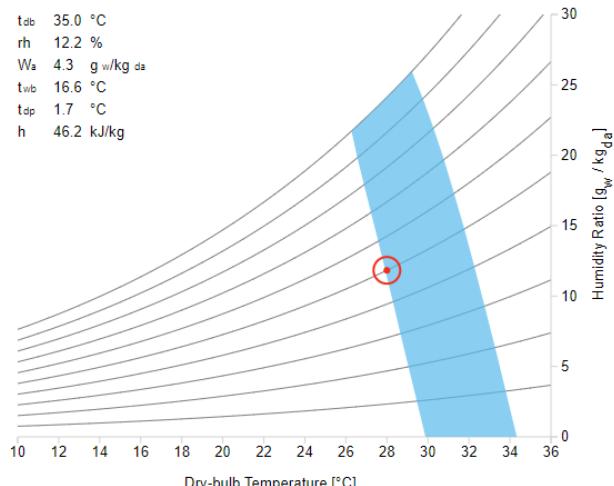
Metabolic rate

1   met

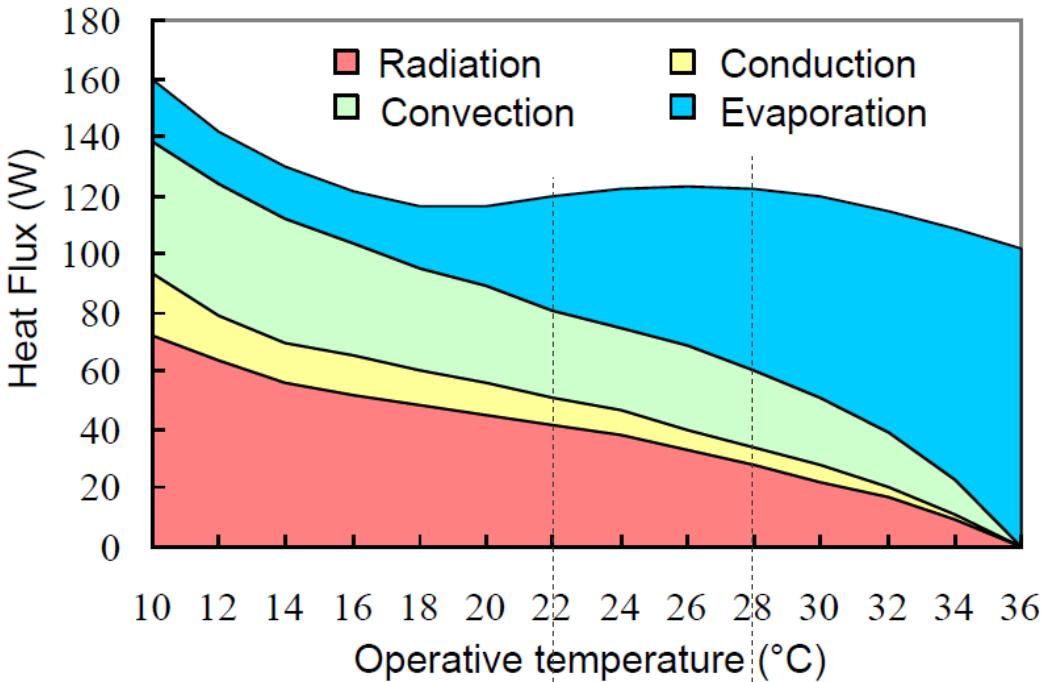
Clothing level

0.5   clo

$t_{db}$  35.0 °C  
 $rh$  12.2 %  
 $W_a$  4.3 g  $w/kg$   $da$   
 $t_{wb}$  16.6 °C  
 $t_{dp}$  1.7 °C  
 $h$  46.2 kJ/kg

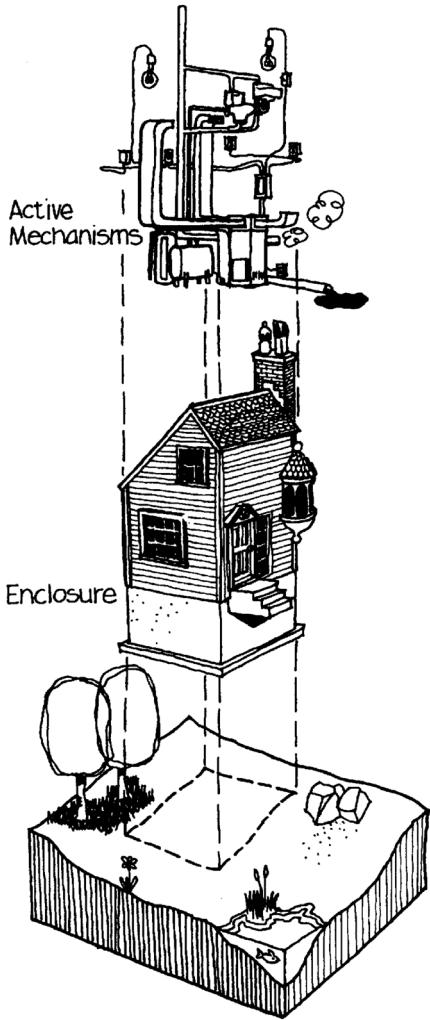


From Lect. 03:



**Radiant heating** is more efficient at low temperatures

**Convective cooling** is more efficient at high temperatures

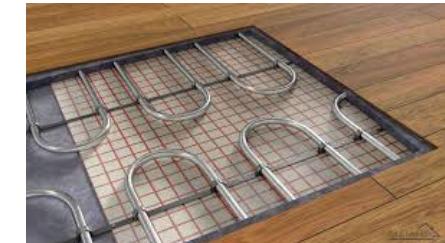
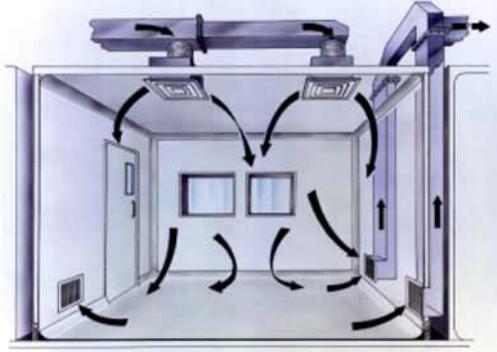


Source: Edward Allen «How Buildings Work» (2005)

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From Week 11

**Radiant systems****centralized** water conditioning**Radiant ceiling****Floor/Wall systems****All-Air Systems****centralized** air conditioning

- Air supply through **grills** (no air deflection)



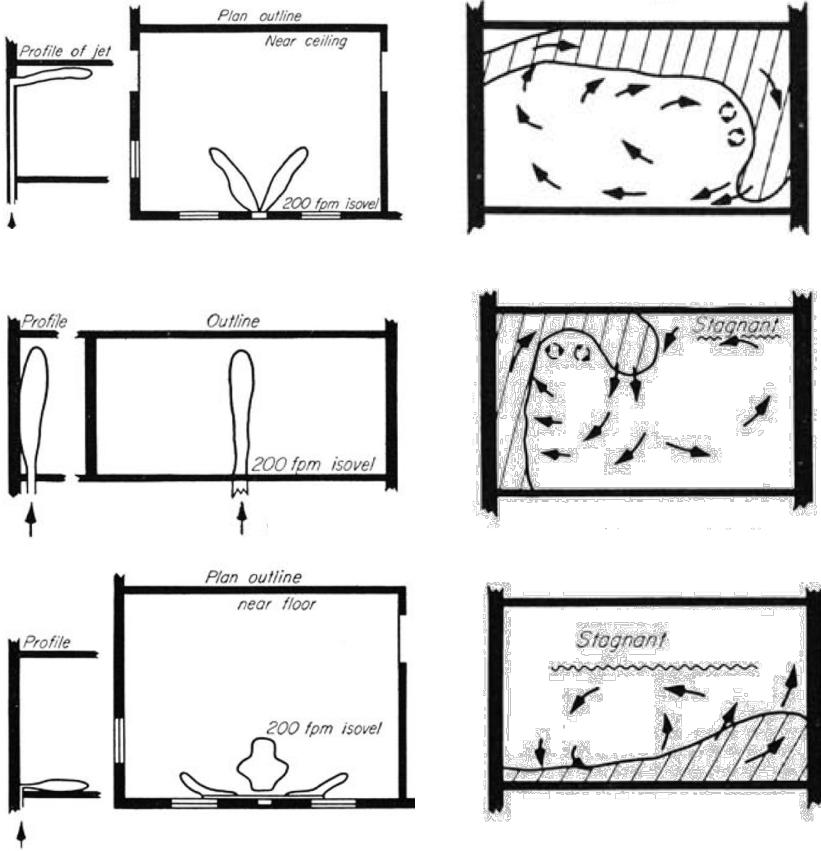
- Air supply through **diffusers** (diffuses air)

**localized** air conditioningheat extraction using: **refrigerant, water****Fancoils, indoor units** (air recirculation)**Chilled beams** (active and passive) thermal conditioning + fresh air supply (active only)

# Convective (All-Air) Cooling: Airflow

## Distinct groups of air patterns:

- Air is discharged *at the ceiling level*
- Air is discharged *vertically* from floor grill (non-spreading floor outlet)
- Air spread *horizontally across the floor* (long baseboard units that spread air over a wide floor area)



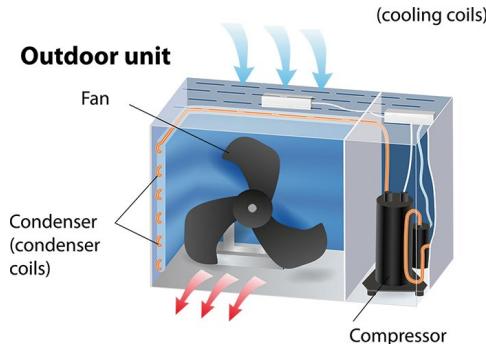
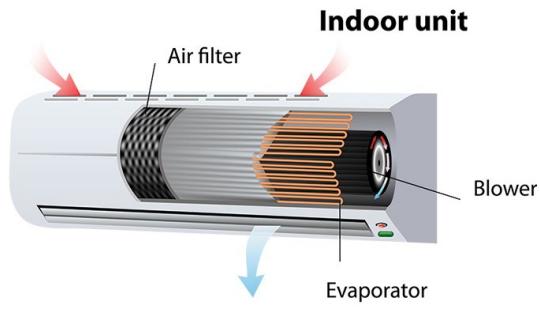
Source: H Straub (1956)

# Split System

- **A split system** is *the most common cooling equipment* that serves **a single zone** (room)
- Indoor air cooling is based on **the vapor-compression cycle**.

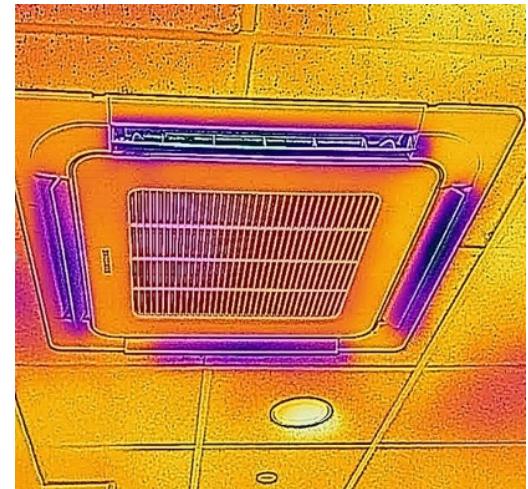
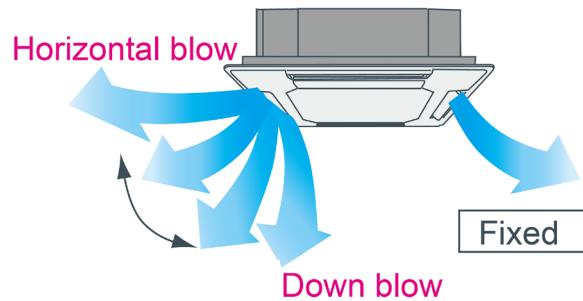
Each split system has 2 units:

- **indoor unit** (an evaporator in a casing, with a fan blower)
- **outdoor unit** (a condenser with a fan, and a compressor in a casing)



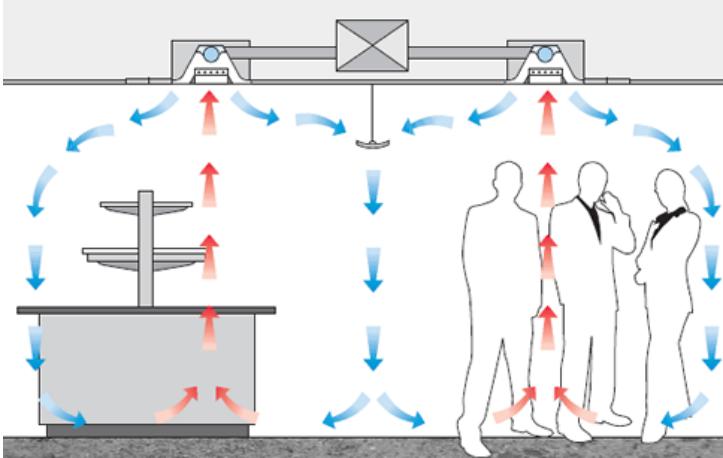
# Fancoils

- Indoor terminal units that **remove heat from the space locally mainly due to the forced convection**
  - a room air *is cooled* by passing through a *fin-and-coil heat exchanger* that is filled with *circulating cold refrigerant*, a *fan* is used to force the airflow through the coil
- They **recirculate indoor air** (fresh air is provided by the separate ventilation system)
- A **fancoil is an evaporator of the vapor-compression cycle** that is used to “generate” cooling
- There can be multiple designs that can fit different space types, cold air is typically directed towards occupied zone:

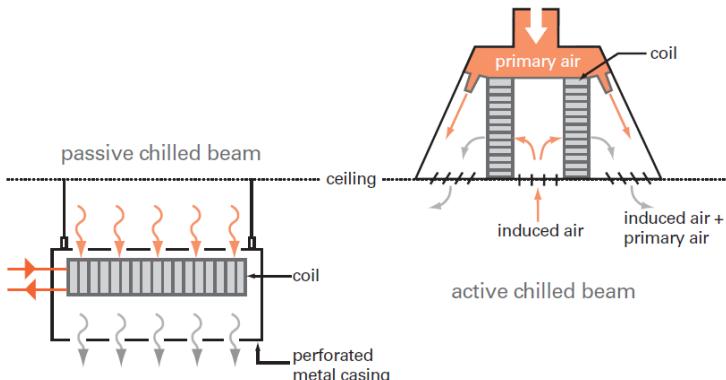


# Chilled Beams

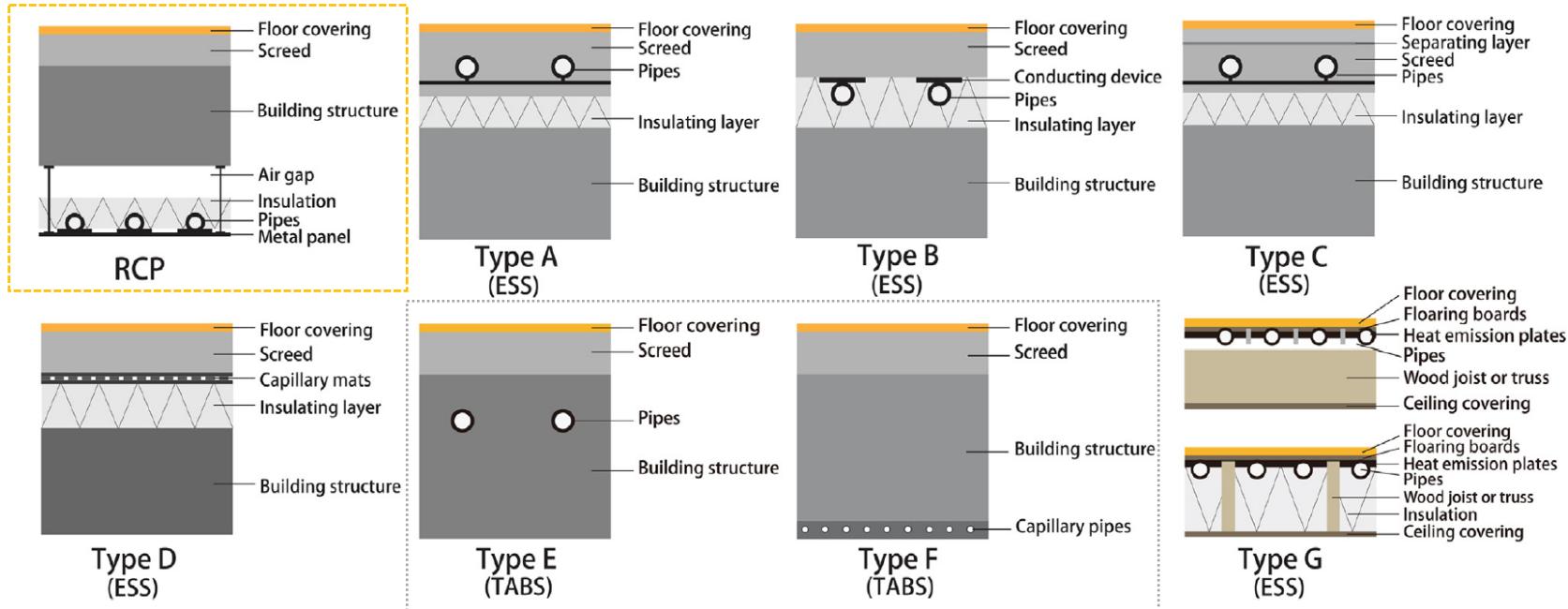
- Indoor terminal units that **remove heat from the space** mainly due to **convection**
- There are **2 types** of **chilled beams**:
  - passive** (no fresh air supply, airflow circulation based on the buoyancy effect)
  - active** (with integral air supply through nozzles)
- Both types have a *fin-and-tube* type *heat exchanger* which provides *air cooling* by removing heat using circulating cool water in pipes.
- The purpose of the **primary air system** for **active chilled beams** is to:
  - Deliver *at least* the required amount of outdoor air to each space **for ventilation**
  - Deliver **air** which is *dry enough* to **offset the space latent load** and maintain the **indoor dew point low enough to avoid condensation** on the chilled beams
  - Deliver **enough air** to **induce sufficient room airflow** to **offset the space sensible cooling load**.

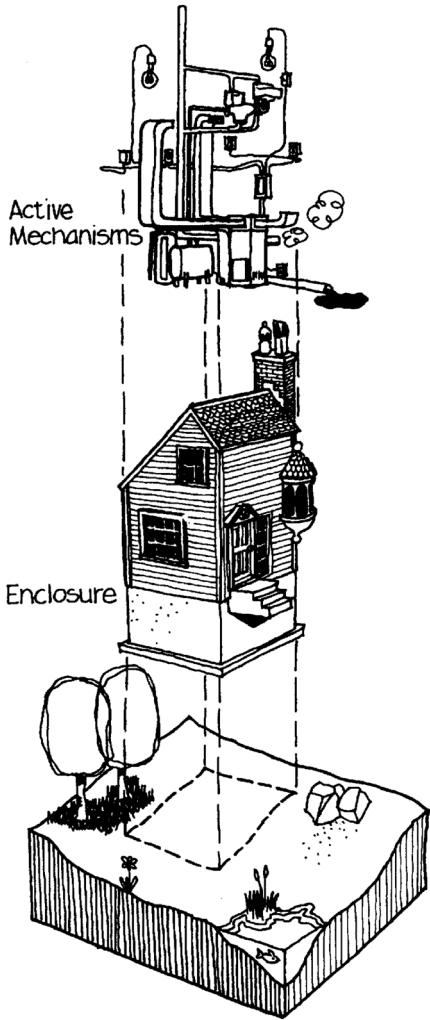


Source: TRANE, Engineering Newsletter, V38-4



- Three main categories of hydronic systems:
  - **radiant ceiling panels (RCP)**,
  - **embedded surface systems (ESS)**
  - **thermally activated building systems (TABS)**





Source: Edward Allen «How Buildings Work» (2005)

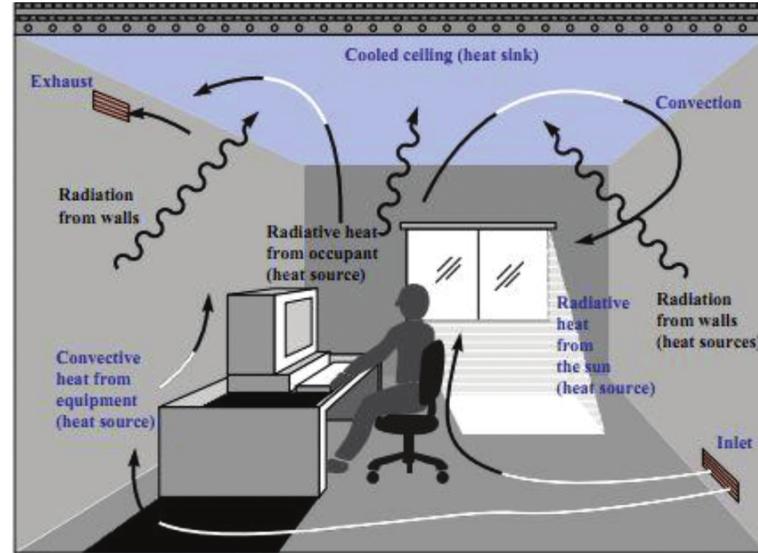
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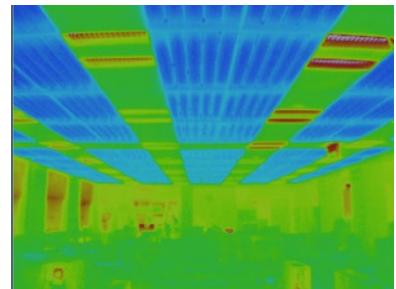
# All-Air Cooling vs. Radiant Cooling

- **Cooling** (rate of sensible and latent energy removal/extraction) required to maintain an indoor environment at a desired temperature and humidity
- **Heat extraction rate** – the rate at which heat is removed from a **space** by the **emission system** (terminal device)
  - **For an all-air system:** the *enthalpy difference* between airflow supplied to the space and airflow *leaving* the space
  - **For a radiant system:** the *sum of convective and radiant heat transfer rates* at the indoor face of internally cooled surface

Source: Woolley et al. (2018) *Energy & Buildings* 176, pp 139–150



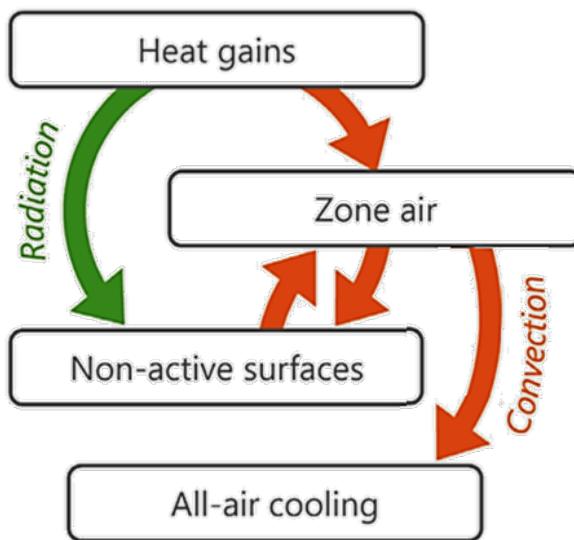
Source: İzzet Yüksek



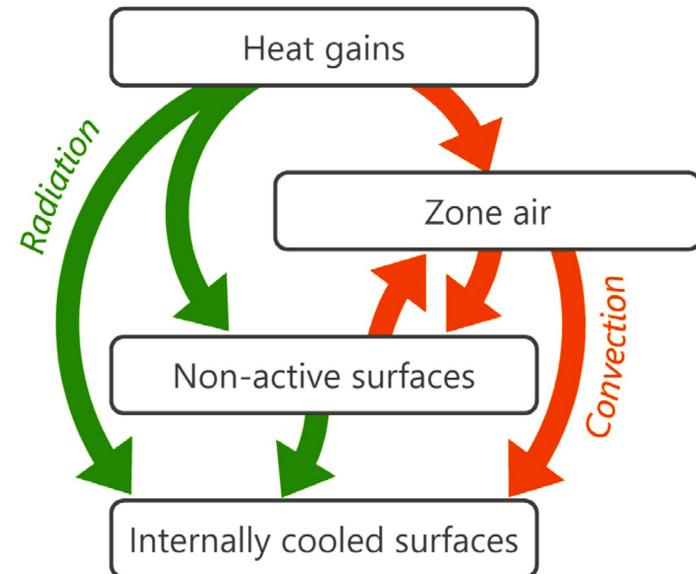
Source: SANKEN

# Heat Transfer Pathways for Cooling

- In a space with **all-air cooling**, all radiant heat gains are absorbed by *non-active surfaces*.
- In a space with **radiant cooling**, a portion of the *radiant heat gains* is absorbed by *non-active surfaces* and a portion is absorbed by the *internally cooled surfaces*.

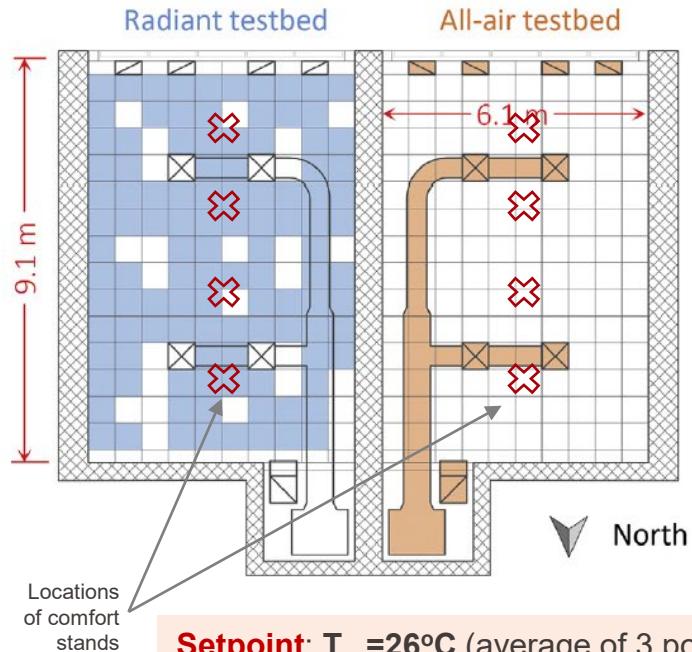


(a) all air cooling



(b) radiant surface cooling

# All-Air Cooling vs. Radiant Ceiling Cooling



**Setpoint:  $T_{op}=26^{\circ}\text{C}$**  (average of 3 points at 0.6m height: 3.45 m, 5.3 m and 7.16 m from the south wall)

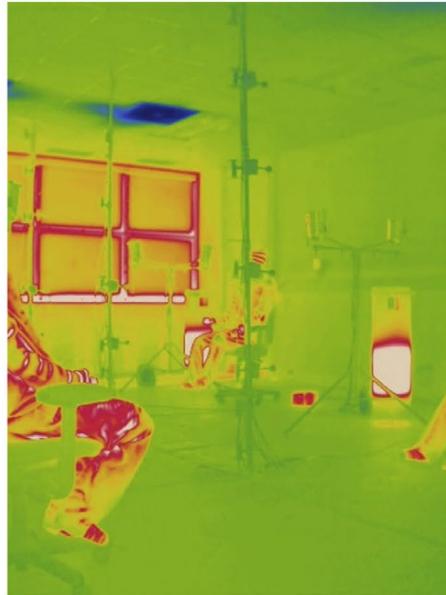
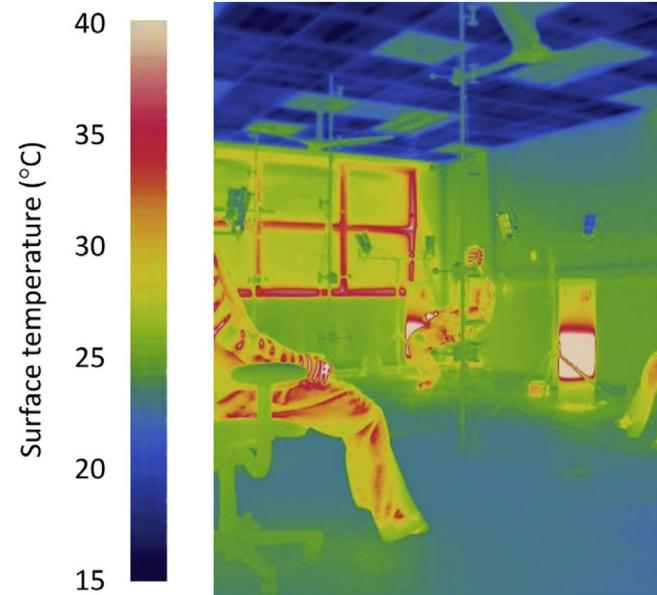
Equal comfort conditions

Radiant	26 °C ≈ 26.5 °C ≈ 25.5 °C	Operative temperature Air temperature Mean radiant temp.	26 °C ≈ 25.5 °C ≈ 26.5 °C	All-air
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Source: Woolley et al. (2018) Energy & Buildings 176, pp 139–150

# All-Air Cooling vs. Radiant Ceiling Cooling



Infrared image of  
radiant testbed

Infrared image of  
all-air testbed

Equal comfort conditions

Radiant

26 °C  
≈ 26.5 °C  
≈ 25.5 °C

Operative temperature  
Air temperature  
Mean radiant temp.

26 °C  
≈ 25.5 °C  
≈ 26.5 °C

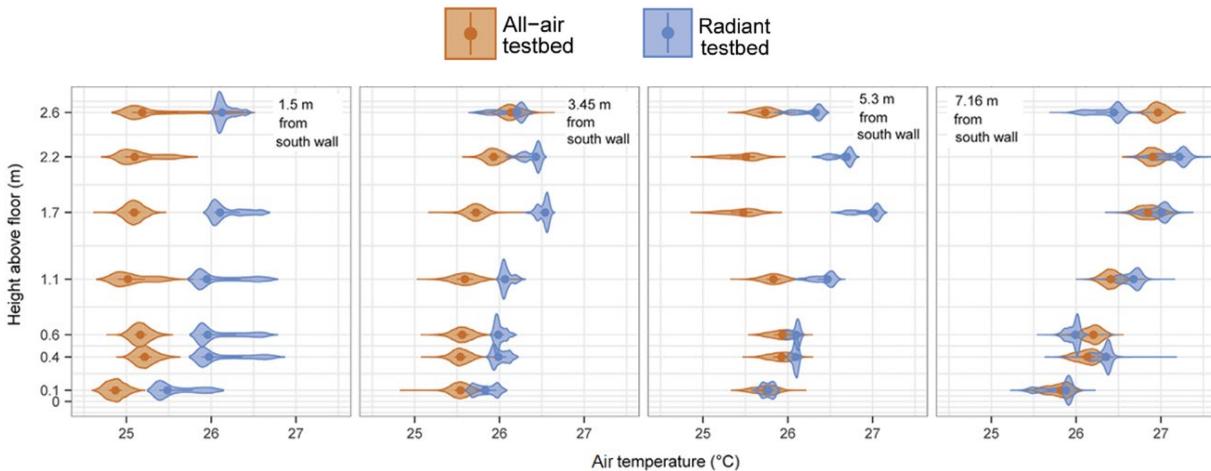
All-air

Environmental conditions

Outdoor air temp. ≈ 30 °C  
Exterior surface temp. ≈ 60 °C  
11:00-13:00@ 7 Sept. 2016

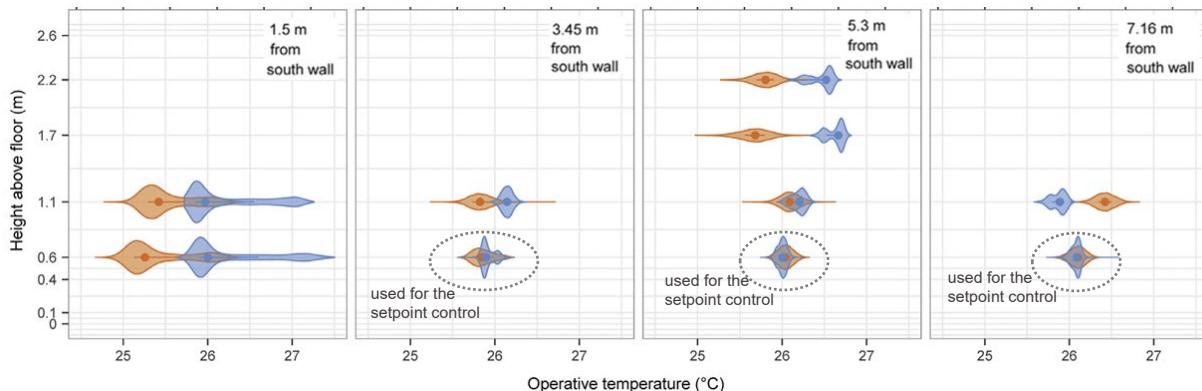
- Air Temperature:

air temperature is higher in a radiant cooled case than in all-air cooled case at equivalent comfort conditions → radiant cooling could reduce heat gain from ventilation if supplied air is warmer.

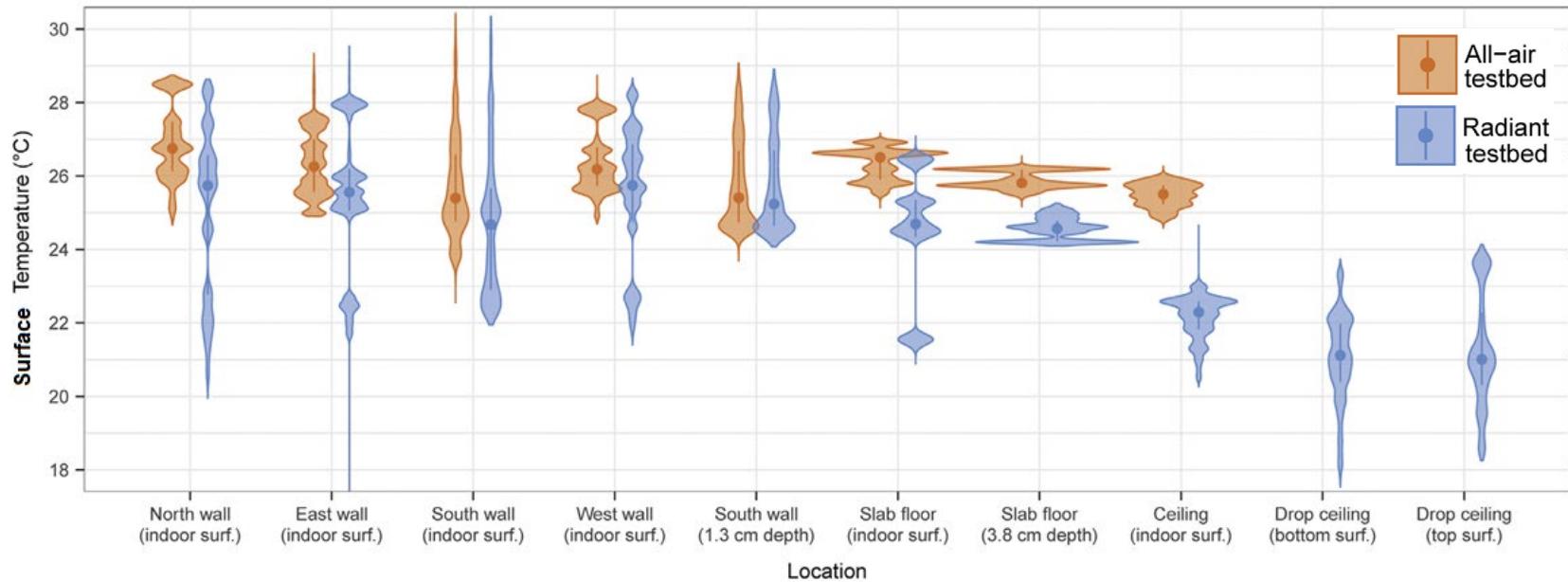


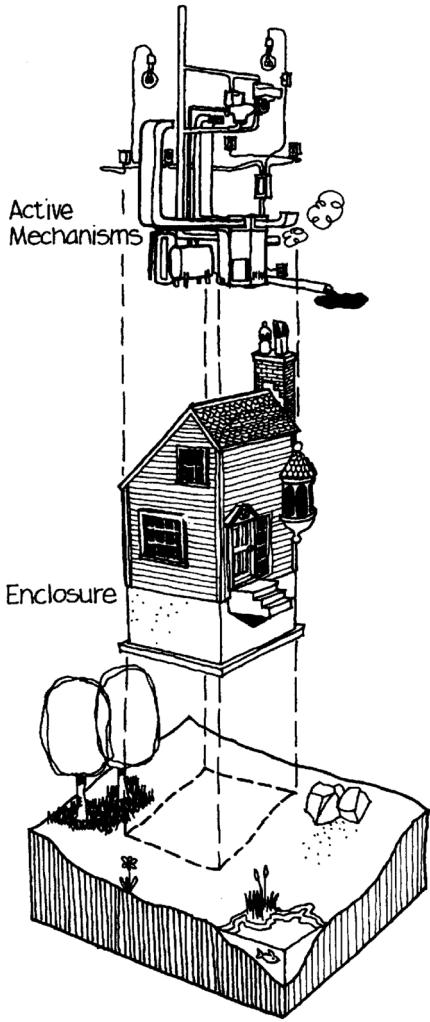
- Operative Temperature:

variation throughout a room (in non-controlled locations) is a consequence of non-uniform distribution of heat gains and distribution of cooling



- **Surface Temperature:** All interior surfaces in a space with radiant cooling are cooler than in a *similar* space with all-air cooling → net heat gain by heat transfer *through the envelope* can be higher





Source: Edward Allen «How Buildings Work» (2005)

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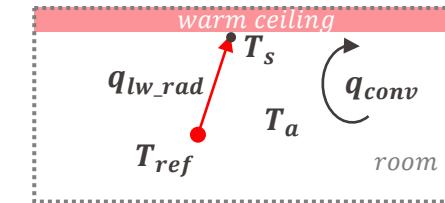
# Embedded Radiant Systems: Rated Heat Flux

- Surface convective heat transfer:
- Longwave radiation heat transfer:
- Total surface heat flux\*:  
(heat flow density,  $\text{W/m}^2$ )

$$q''_{conv} = h_c \times (T_a - T_s)$$

$$q''_{lw\_surf} = h_{rad} \cdot (T_s - T_{ref})$$

$$q'' = h_t \cdot |T_s - T_{ref}|$$



$T_s = T_{m,s}$  mean surface temperature

$T_{ref} = T_{op}$  operative temperature (per ISO 18555)

- **Heat transfer coefficient** between radiant surface and room condition:

Combined convection and (longwave) radiation heat transfer coefficient  $h_t$ :

Heating:		Cooling	
Floor heating (FH)	11 $\text{W/m}^2\text{K}$	Floor cooling (FC)	7 $\text{W/m}^2\text{K}$
Wall heating (WH)	8 $\text{W/m}^2\text{K}$	Wall cooling (WC)	8 $\text{W/m}^2\text{K}$
Ceiling heating (CH)	6 $\text{W/m}^2\text{K}$	Ceiling cooling (CC)	11 $\text{W/m}^2\text{K}$

Radiant (longwave) heat transfer coefficient  $h_{rad} = 5.5 \text{ W/m}^2\text{K}$   
(less than 4% error for surface temperatures in the range of 15-30°C)

- **Heat flow density ( $\text{W/m}^2$ ):** at steady-state

- **floor heating / ceiling cooling:**  $q = 11 \cdot |T_{m,s} - T_{op}|$
- **wall heating / wall cooling:**  $q = 8 \cdot |T_{m,s} - T_{op}|$
- **ceiling heating:**  $q = 6 \cdot |T_{m,s} - T_{op}|$
- **floor cooling:**  $q = 7 \cdot |T_{m,s} - T_{op}|$

\* Shortwave radiation heat transfer is not considered in the standard approach of defining total heat flow density which might become problematic for floor cooling with direct sunlight

What is the **comfort-related limiting heat flux** for **CEILING cooling** in the case of **indoor operative temperature 26°C** if the **surface temperature limit is 17°C**?

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- A. 100 W/m<sup>2</sup>
- B. 42 W/m<sup>2</sup>
- C. 72 W/m<sup>2</sup>
- D. None above

# What is the **comfort-related limiting heat flux** for **FLOOR cooling** in the case of **indoor operative temperature 26°C** if the **surface temperature limit is 20°C**?

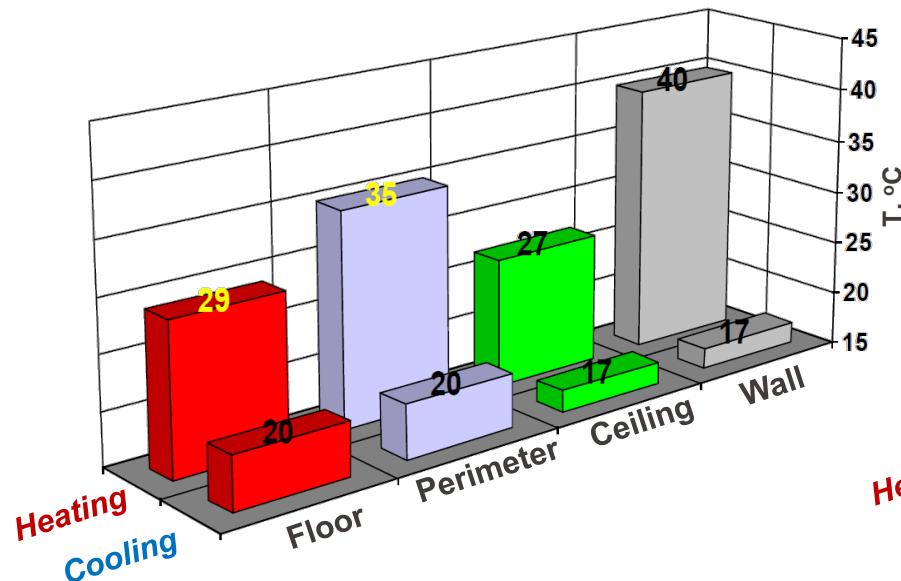
- A. 100 W/m<sup>2</sup>
- B. 42 W/m<sup>2</sup>
- C. 72 W/m<sup>2</sup>
- D. None above

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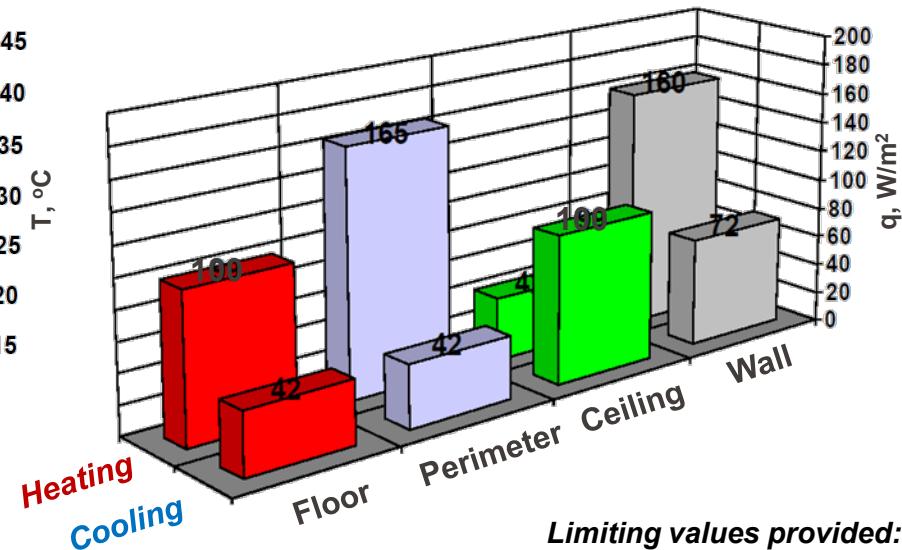
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# Radiant Systems: Maximum Heat Output

- **Maximum** heating and cooling capacity depends *on the position* of the radiant surface
- The heating/cooling capacity *is limited* by **relatively small convective heat transfer coefficient** between *the surface and air*, risk of condensation and concerns about surface temperature, radiant asymmetry, vertical air stratification and draft from comfort perspective

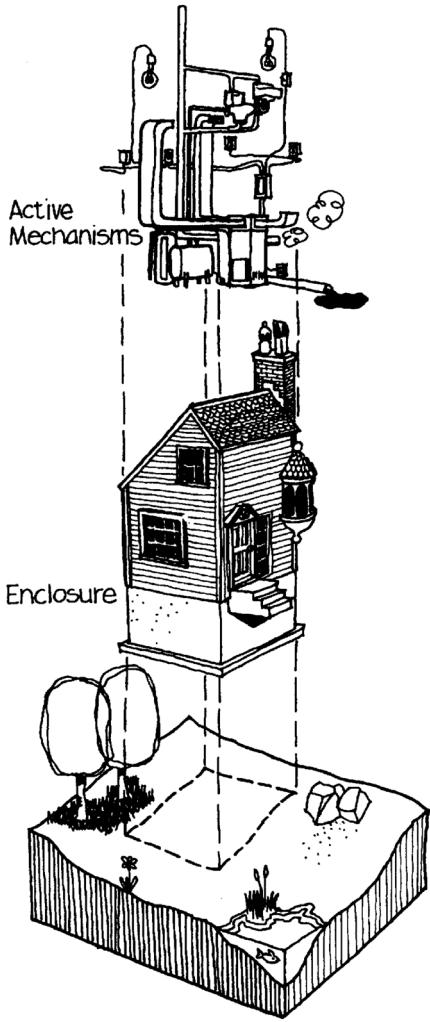


Source: Olesen (2018), Radiant heating/cooling dimensioning



**Limiting values provided:**

- for **heating**:  $T_{\text{op}}=20^{\circ}\text{C}$
- for **cooling**:  $T_{\text{op}}=26^{\circ}\text{C}$   
(for  $V_{\text{air}} < 0.2 \text{ m/s}$ )



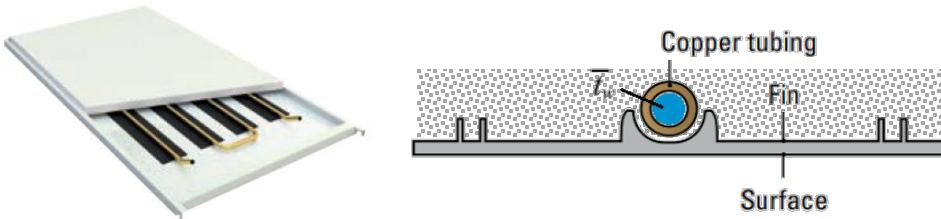
Source: Edward Allen «How Buildings Work» (2005)

# CONTENT:

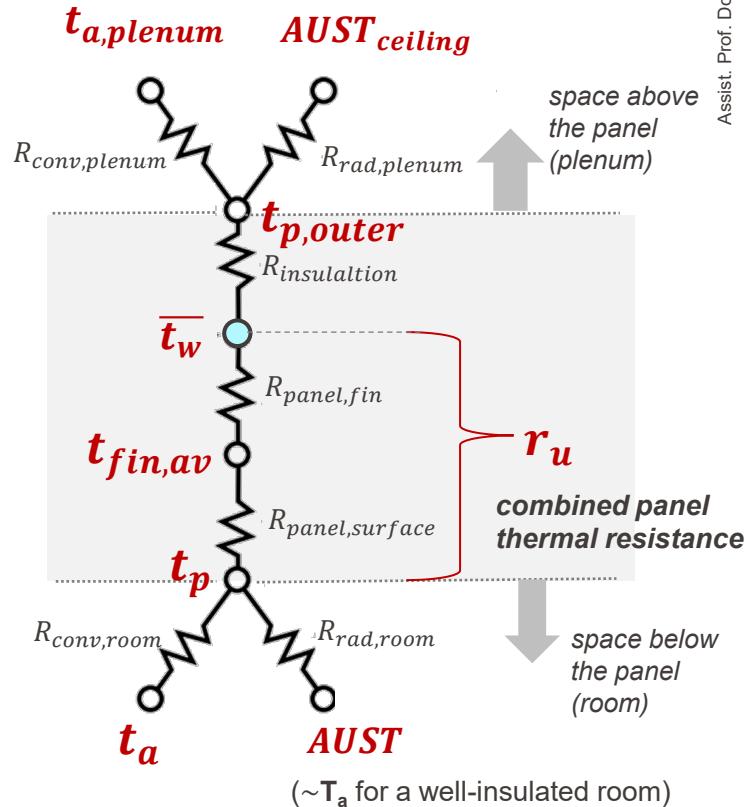
- **Introduction**
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# Radiant Systems: Ceiling Panels

- A radiant ceiling is a large heat exchanger suspended from a room's ceiling



- Thermal resistance in the panel to heat transfer from or to its surface reduces the performance of the system by affecting water temperature ( $\bar{t}_w$ )
- Average water temperature:  $\bar{t}_w = (T_{wi} + T_{wo})/2$
- Combined thermal resistance of the panel ( $r_u$ ) considers architecture of the panel (type of the bond between the piping and the panel material, spacing, piping, covering material presence, etc.)
  - When the back side of the panels is insulated using an insulation material, the heat exchange toward the plenum could be ignored



Thermal resistance of panels ( $m^2 \cdot K$ )/W:

$$r_u = r_t \cdot M + r_s \cdot M + r_p + r_c$$

- $r_u$  - combined panel thermal resistance for a given spacing  $M$  of adjacent tubes
- $r_t$  - thermal resistance of tube wall per unit tube spacing:

Circular pipes:

$$r_t = \frac{\ln(D_o/D_i)}{2\pi k_t}$$

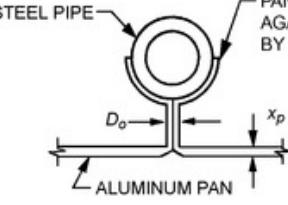
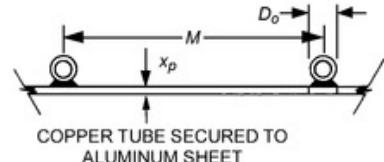
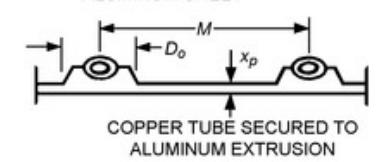
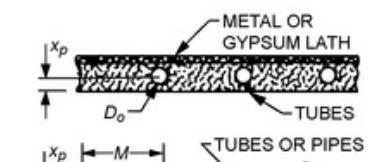
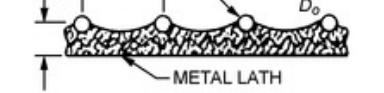
Elliptical pipes:

$$r_t = \ln \frac{(a_o + b_o)/(a_i + b_i)}{2\pi k_t}$$

- $r_s$  - thermal resistance between tube and panel body per unit spacing between adjacent tubes (see the tabulated values)
- $r_p$  - thermal resistance of a panel body (see the table)
- $r_c$  - thermal resistance of the active panel surface covering (i.e., pads, carpets, etc.), for thickness of  $x_c$ :

$$r_c = x_c/k_c$$

- $k_t, k_p, k_c$  - thermal conductivity of tubing, panel, and covering materials,  $W/(m \cdot K)$

Type of Panel	$r_p$	$r_s$	Thermal Resistance
 STEEL PIPE ALUMINUM PAN $D_o$ $x_p$	$\frac{x_p}{k_p}$	0.32	
 COPPER TUBE SECURED TO ALUMINUM SHEET $D_o$ $x_p$ $M$	$\frac{x_p}{k_p}$	0.38	
 COPPER TUBE SECURED TO ALUMINUM EXTRUSION $D_o$ $x_p$ $M$	$\frac{x_p}{k_p}$	0.1	
 METAL OR GYPSUM LATH TUBES $x_p$ $D_o$ $M$	$\frac{x_p - D_o/2}{k_p}$	$\approx 0$	
 TUBES OR PIPES $x_p$ $M$ $D_o$ METAL LATH	$\frac{x_p - D_o/2}{k_p}$	$\leq 0.12$	

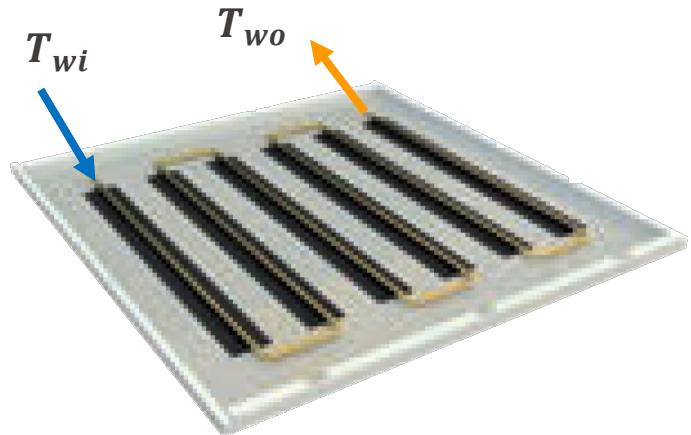
Source: ASHRAE Handbook 2020, HVAC Systems and Equipment, Chapter 6

# Surface Cooling: Condensation Issues

- To prevent condensation on the room side of cooling panels, the panel water supply temperature ( $T_{wi}$ ) should be maintained at least 1 K above the room design dew-point temperature ( $T_{dp}$ ):

$$(T_{wi} - T_{dp}) > 1K$$

- This minimum difference is recommended to allow for the normal drift of temperature controls for the water and air systems, and also to provide a factor of safety for temporary increase in space humidity.
- Precooling and dehumidification of outdoor fresh air before it gets mixed up with the return air from the air-conditioned space, is recommended to prevent the surface condensation at panel surface

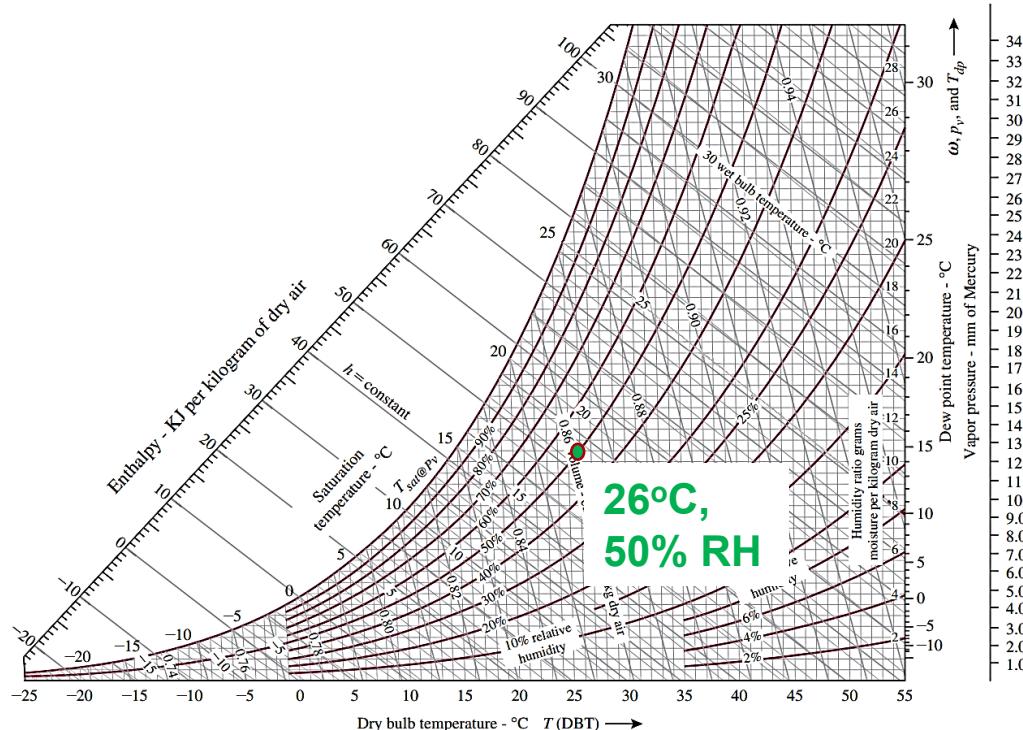


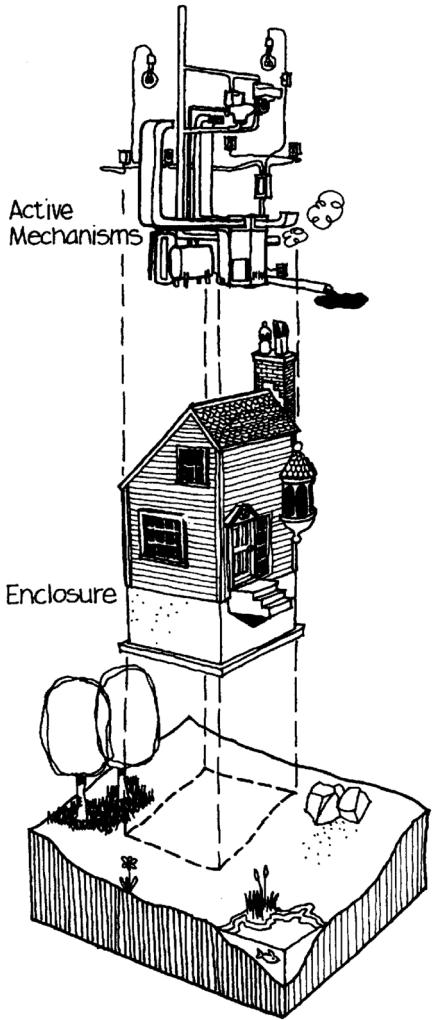
# What is the dew point temperature when dry-bulb temperature is 26°C and relative humidity is 50%?

- A. 13.8
- B. 14.8
- C. 15.0
- D. 26.0

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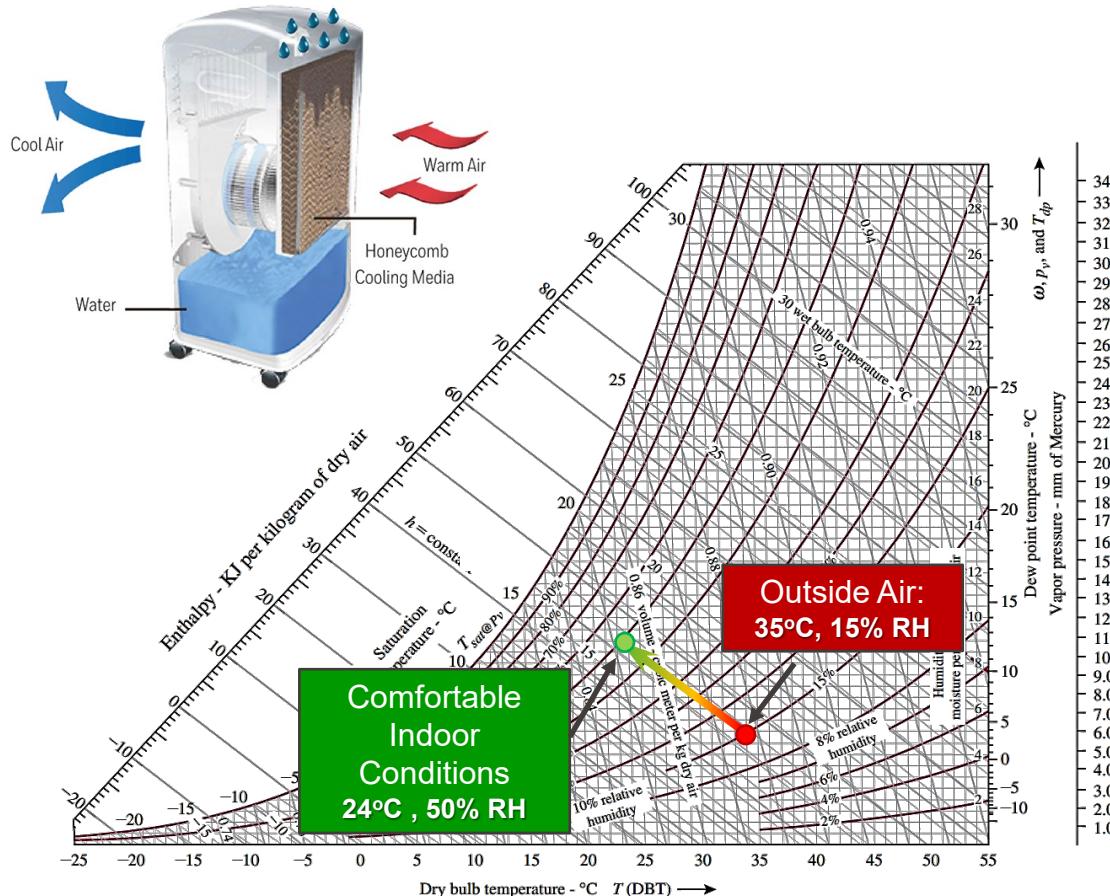
Source: Edward Allen «How Buildings Work» (2005)

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# Evaporative Cooling

- **Simultaneous heat and mass transfer** process, or **adiabatic cooling**, and follows a **constant enthalpy line** on the psychrometric chart.
- The *cooling potential* for evaporative cooling is dependent on **the wet-bulb depression** (difference between the **dry-bulb** temperature and the **wet-bulb** temperature)
- Application is limited to dry locations



**How much can air temperature drop if evaporative cooling is used for the following conditions:**

- Ambient temperature: **28°C, 40% RH**
- Maximum desired RH is **60%**

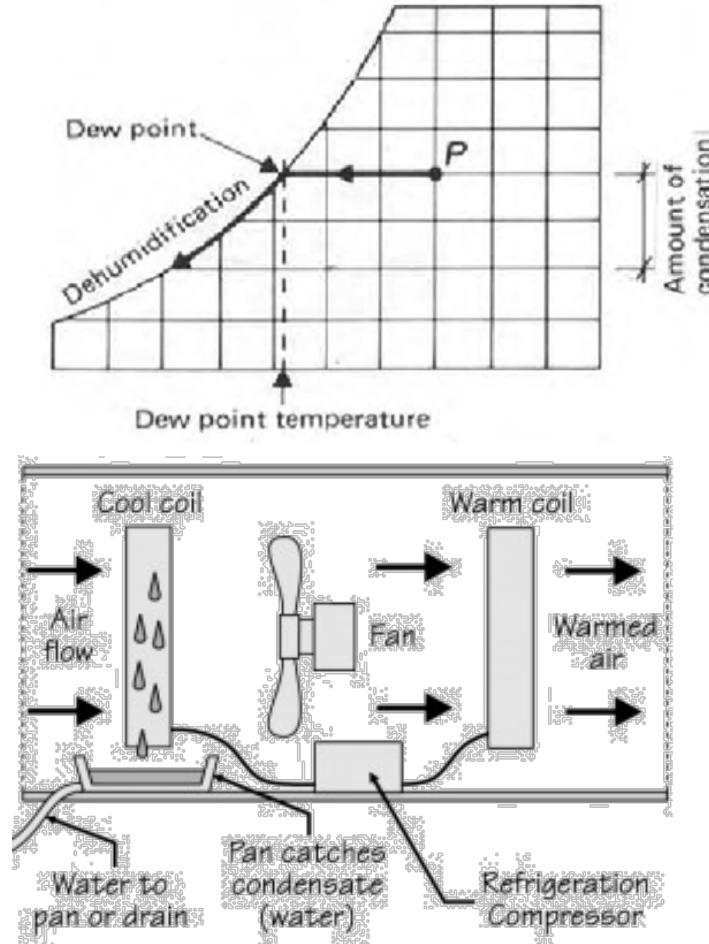
- A. 1°C**
- B. 2°C**
- C. 3°C**
- D. 4°C**

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# Dehumidification

- **Dehumidification** – extraction of the water vapor that is contained in air
- *The most common method of dehumidification: sub-cooling of air in cooling coils to condense out moisture using chilled water or refrigerant*
- **Alternative strategies that** could be more energy efficient for certain climate zones and application types:
  - desiccant dehumidification (solid, liquid)
  - liquid desiccant dehumidification
  - dual wheel, wrapped around coil, etc.
- *The pre-cooling and dehumidification of outdoor fresh air can be incorporated into general air handling units (AHU).*

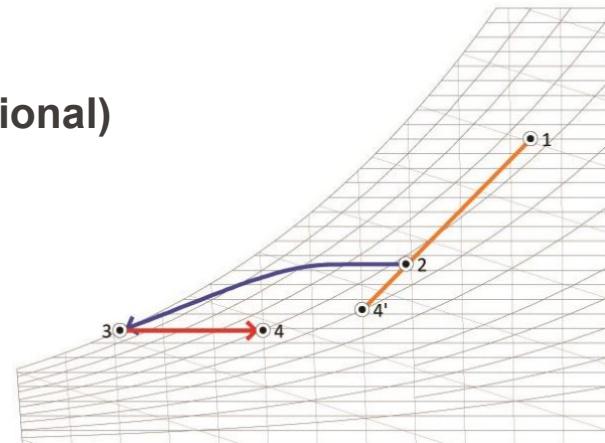
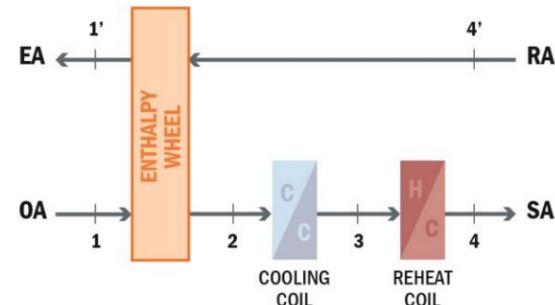


- Chilled water cooling/ dehumidification (conventional)

1-2: heat recovery (energy exchange between the used air and entering fresh air)

2-3: air passes over the cooling coil to cool down to the required air humidity ratio

3-4: reheating to the required supply air temperature



Assist. Prof. Dolaana Kholvaly

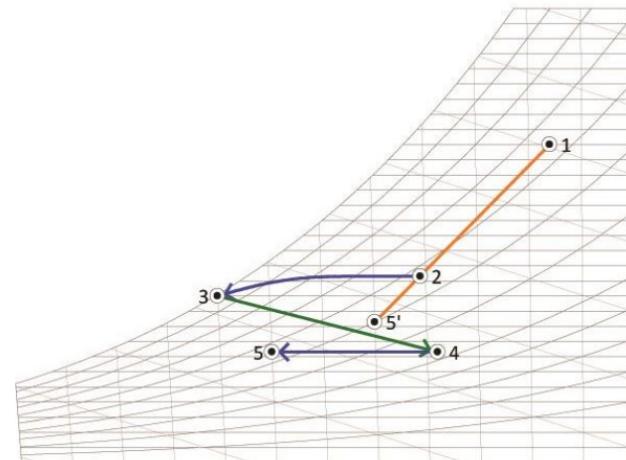
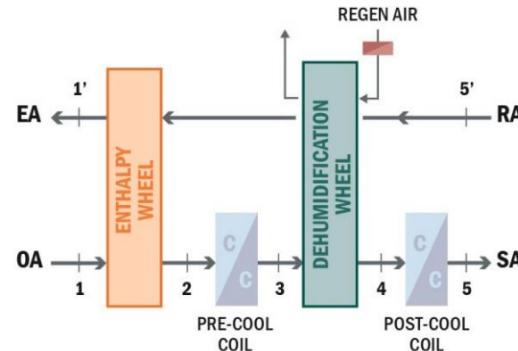
- Solid desiccant dehumidification

1-2: heat recovery

2-3: air temperature reduction

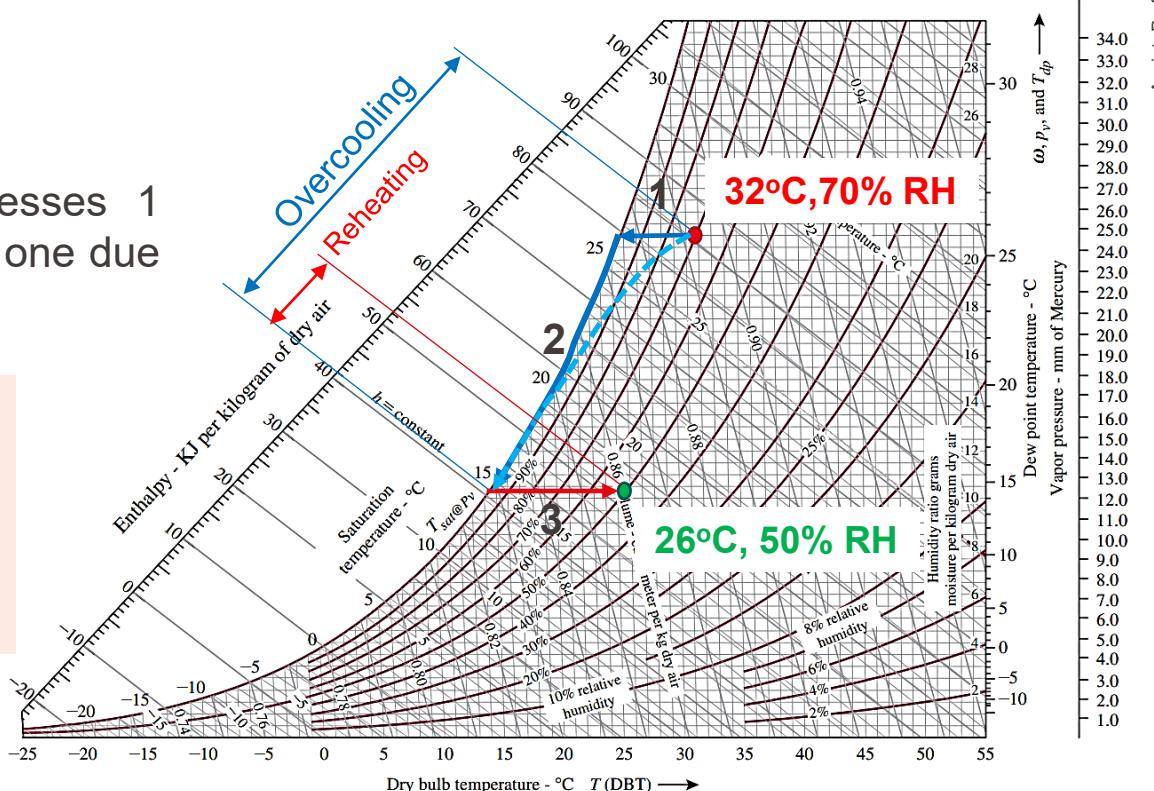
3-4: isenthalpic dehumidification, the moisture content of the air reduces while air temperature increases

4-5: warm air passes over a post-cooling coil to cool it down to the required supply air temperature

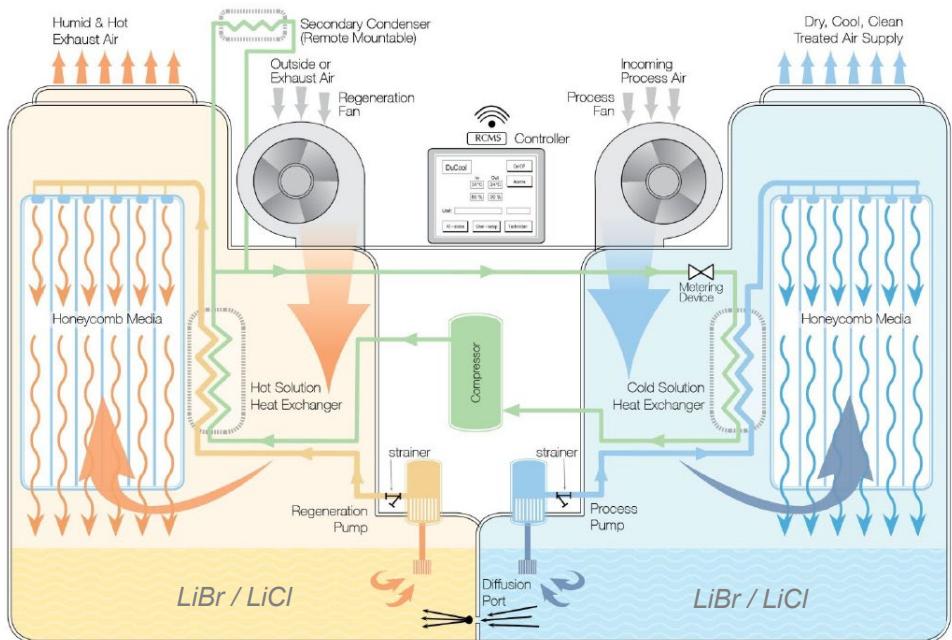
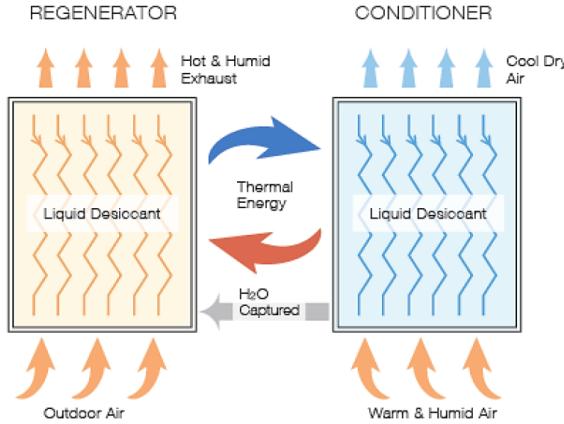


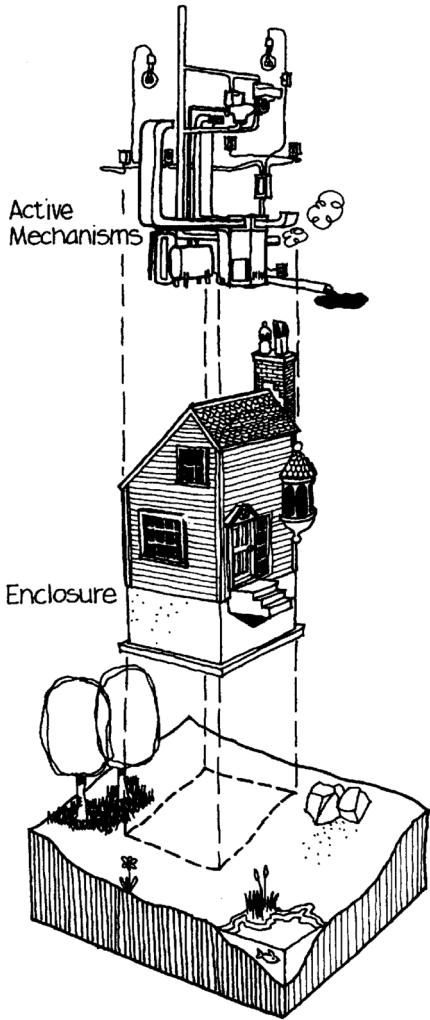
- Conventional dehumidification follows 3 processes:
  - Sensible cooling**
  - Latent cooling**
  - Sensible heating**
- In actual process, processes 1 and 2 deviate from ideal one due to the flow cooling.

What is the *alternative more efficient process path* for the same given indoor and outdoor conditions?



- Liquid desiccant dehumidification





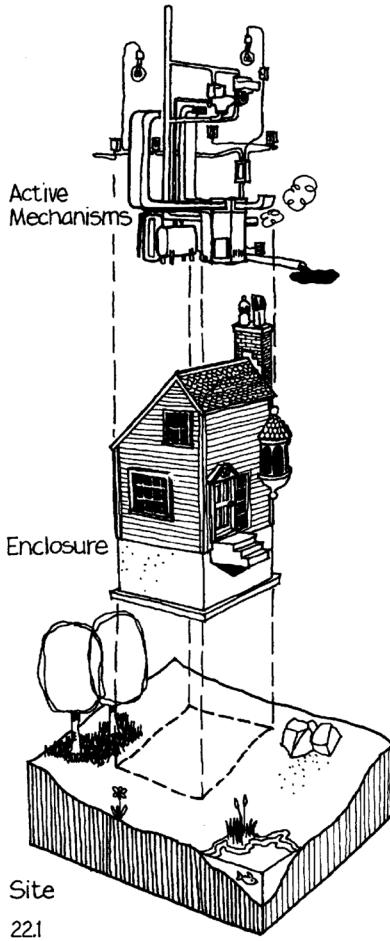
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- **Passive methods to keep buildings cool**  
Preventing, modulating, and dissipating heat
- **Mechanical cooling emission systems categorization**
  - **All-Air Systems:** flow patterns depending on the supply direction
  - **Refrigerant-based systems:** split system, fancoils, chilled beams
  - **Hydronic Radiant Systems:** radiant ceiling panels
- **Radiant cooling:** heat transfer pathways, effect on the indoor environment, limiting heat flux, thermal resistance of radiant ceiling panels, condensation consideration
- **Evaporative cooling basics**
- **Dehumidification:** differences between conventional approach, solid vs. liquid desiccant dehumidification





Source: Edward Allen «How Buildings Work» (2005)

**Thank you  
for your attention!**

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